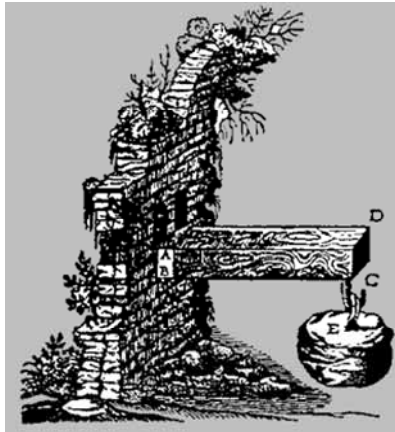


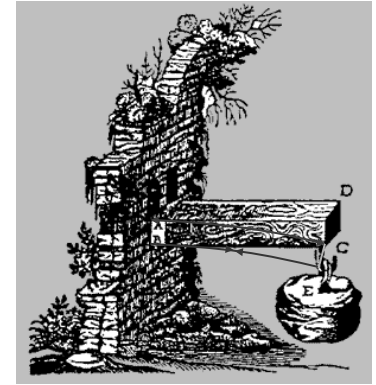
lecture
 eighteen

beams:
 bending and shear



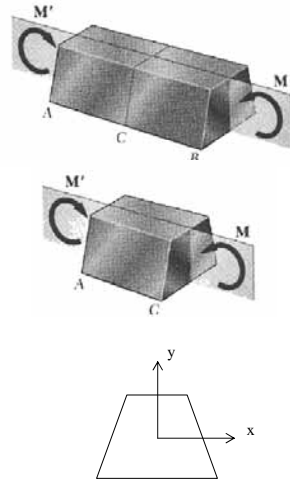
Beam Bending

- Galileo
 - relationship between stress and depth²
- can see
 - top squishing
 - bottom stretching
- what are the stress across the section?



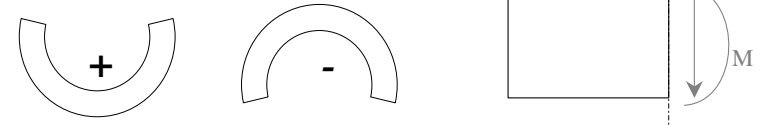
Pure Bending

- bending only
- no shear
- axial normal stresses from bending can be found in
 - homogeneous materials
 - plane of symmetry
 - follow Hooke's law



Bending Moments

- sign convention:



- size of maximum internal moment will govern our design of the section

Normal Stresses

- **geometric fit**
 - plane sections remain plane
 - stress varies linearly

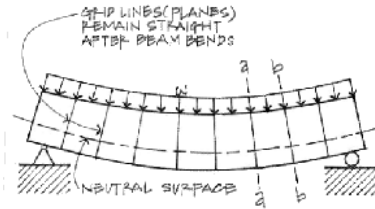


Figure 8.5(b) Beam bending under load.

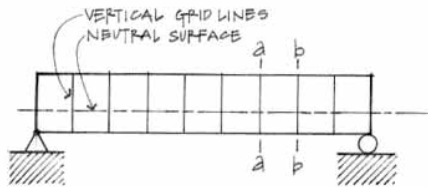
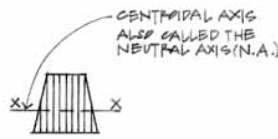


Figure 8.5(a) Beam elevation before loading.



Beam cross section.

Neutral Axis

- stresses vary linearly
- zero stress occurs at the centroid
- neutral axis is line of centroids (n.a.)

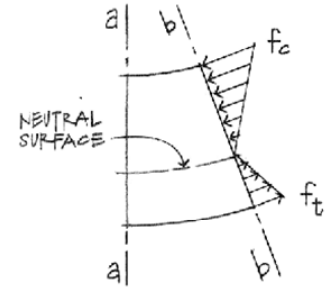


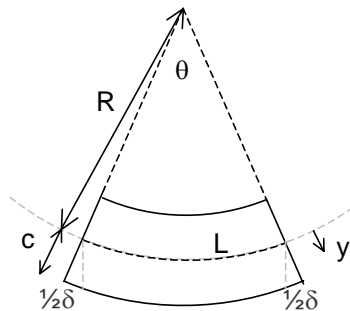
Figure 8.8 Bending stresses on section b-b.

Derivation of Stress from Strain

- pure bending = arc shape

$$L = R\theta$$

$$L_{outside} = (R + y)\theta$$



$$\epsilon = \frac{\delta}{L} = \frac{L_{outside} - L}{L} = \frac{(R + y)\theta - R\theta}{R\theta} = \frac{y}{R}$$

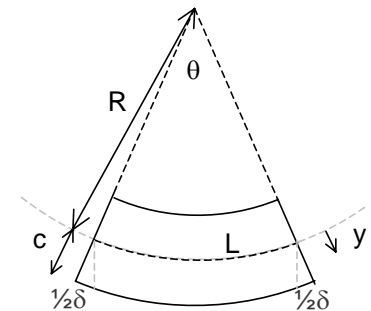
Derivation of Stress

- zero stress at n.a.

$$f = E\epsilon = \frac{Ey}{R}$$

$$f_{max} = \frac{Ec}{R}$$

$$f = \frac{y}{c} f_{max}$$



Bending Moment

- resultant moment from stresses = bending moment!

$$M = \Sigma fy\Delta A$$

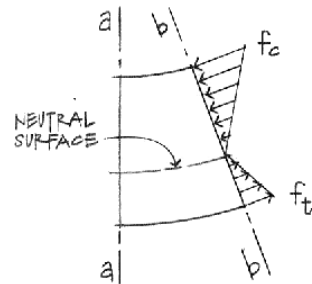


Figure 8.8 Bending stresses on section b-b.

$$= \Sigma \frac{yf_{max}}{c} y\Delta A = \frac{f_{max}}{c} \Sigma y^2 \Delta A = \frac{f_{max}}{c} I = f_{max} S$$

Bending Stress Relations

$$\frac{1}{R} = \frac{M}{EI}$$

curvature

$$f_b = \frac{My}{I}$$

general bending stress

$$S = \frac{I}{c}$$

section modulus

$$f_b = \frac{M}{S}$$

maximum bending stress

$$S_{required} \geq \frac{M}{F_b}$$

required section modulus for design