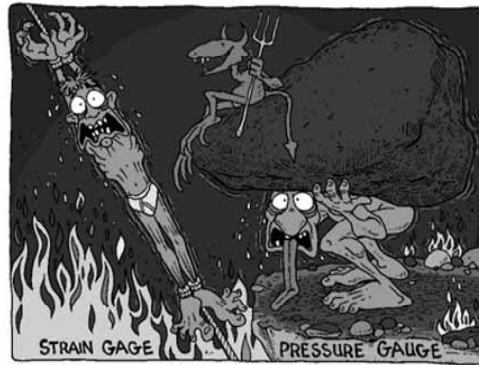


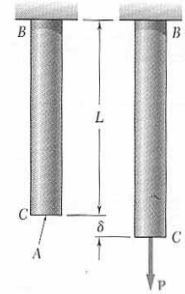
lecture
 sixteen

elasticity
 & strain



Deformations

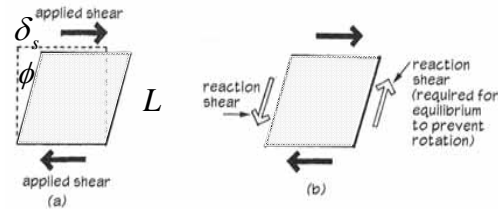
- materials deform
- axially loaded materials change length
- normal stress is load per unit area
- STRAIN:
 - change in length over length
 - UNITLESS



$$\epsilon = \frac{\delta}{L}$$

Shearing Strain

- deformations with shear
- parallelogram
- change in angles
- stress: τ
- strain: γ
 - unitless (radians)

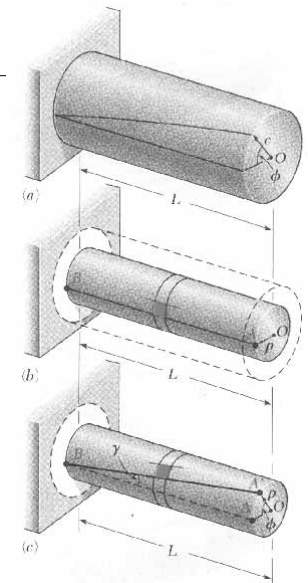


$$\gamma = \frac{\delta_s}{L} = \tan \phi \cong \phi$$

Shearing Strain

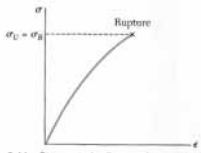
- deformations with torsion
- twist
- change in angle of line
- stress: τ
- strain: γ
 - unitless (radians)

$$\gamma = \frac{\rho\phi}{L}$$



Load and Deformation

- for stress, need P & A
- for strain, need δ & L
 - how?
 - TEST with load and measure
 - plot P/A vs. ϵ



Strain 10
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Material Behavior

- every material has its own response
 - 10,000 psi
 - $L = 10$ in
 - Douglas Fir vs. steel?

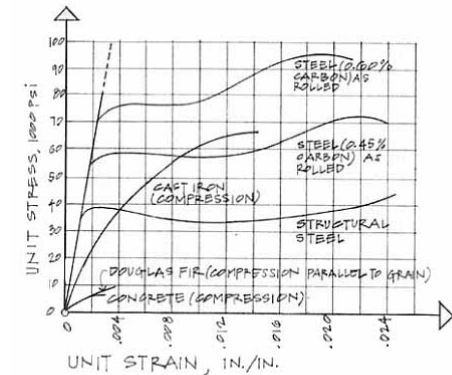


Figure 5.20 Stress-strain diagram for various materials.

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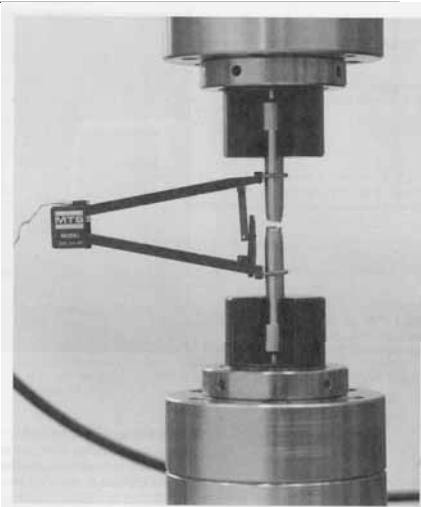
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Behavior Types

- ductile - “necking”
- true stress
- engineering stress
 - (simplified)

$$f = \frac{P}{A}$$

$$f = \frac{P}{A_o}$$



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Behavior Types

- brittle
- semi-brittle

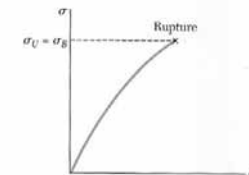


Fig. 2.11 Stress-strain diagram for a typical brittle material.

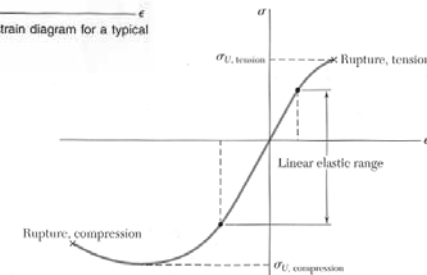


Fig. 2.14 Stress-strain diagram for concrete.

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Stress to Strain

- important to us in f - ϵ diagrams:
 - straight section
 - **LINEAR-ELASTIC**
 - recovers shape (no permanent deformation)

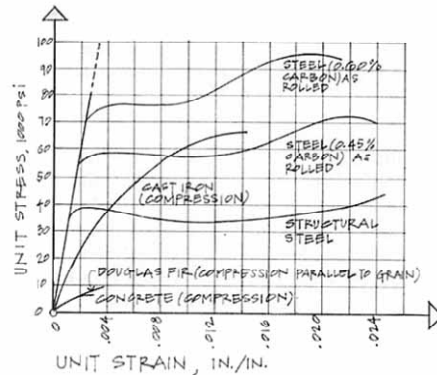
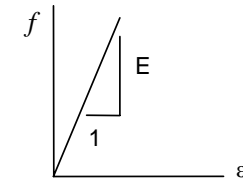


Figure 5.20 Stress-strain diagram for various materials.

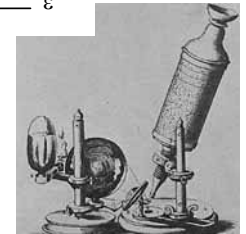
Hooke's Law

- straight line has constant slope
- Hooke's Law

$$f = E \cdot \epsilon$$



- E
 - Modulus of elasticity
 - Young's modulus
 - units just like stress



Stiffness

- ability to resist strain
- steels
 - same E
 - different yield points
 - different ultimate strength

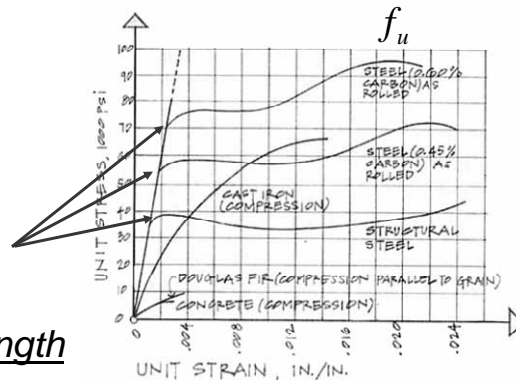
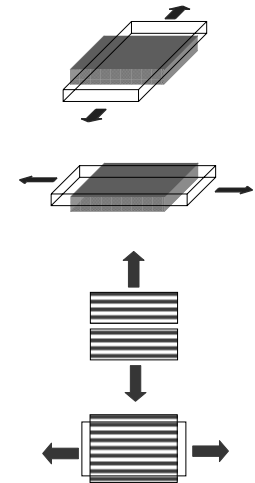


Figure 5.20 Stress-strain diagram for various materials.

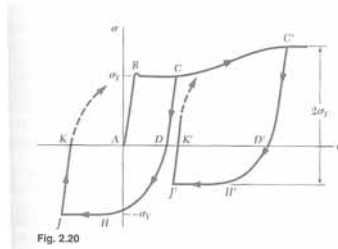
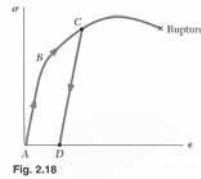
Isotropy & Anisotropy

- **ISOTROPIC**
 - materials with E same at any direction of loading
 - ex. steel
- **ANISOTROPIC**
 - materials with different E at any direction of loading
 - ex. wood is orthotropic



Elastic, Plastic, Fatigue

- elastic springs back
- plastic has permanent deformation
- fatigue caused by reversed loading cycles



Lateral Strain

- or “what happens to the cross section with axial stress”

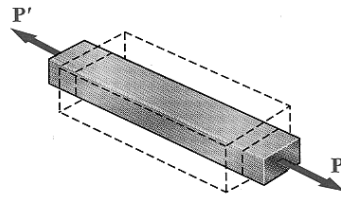
$$\epsilon_x = \frac{f_x}{E}$$

$$f_y = f_z = 0$$

- strain in lateral direction

– negative

– equal for isometric materials



$$\epsilon_y = \epsilon_z$$

Plastic Behavior

- ductile

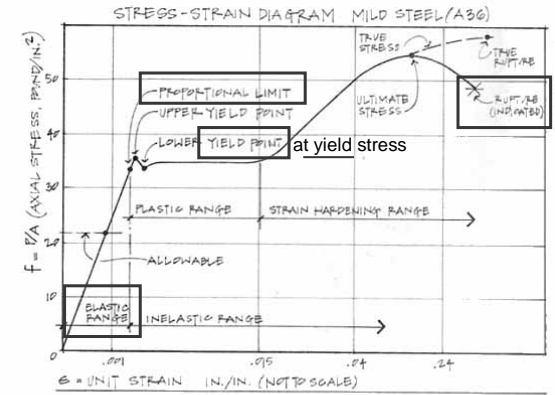


Figure 5.22 Stress-strain diagram for mild steel (A36) with key points highlighted.

Poisson's Ratio

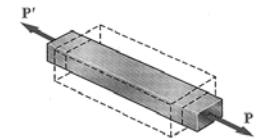
- constant relationship between longitudinal strain and lateral strain

$$\mu = - \frac{\text{lateral strain}}{\text{axial strain}} = - \frac{\epsilon_y}{\epsilon_x} = - \frac{\epsilon_z}{\epsilon_x}$$

$$\epsilon_y = \epsilon_z = - \frac{\mu f_x}{E}$$

- sign!

$$0 < \mu < 0.5$$



Calculating Strain

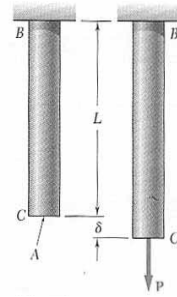
- from Hooke's law

$$f = E \cdot \varepsilon$$

- substitute

$$\frac{P}{A} = E \cdot \frac{\delta}{L}$$

- get \Rightarrow
$$\delta = \frac{PL}{AE}$$



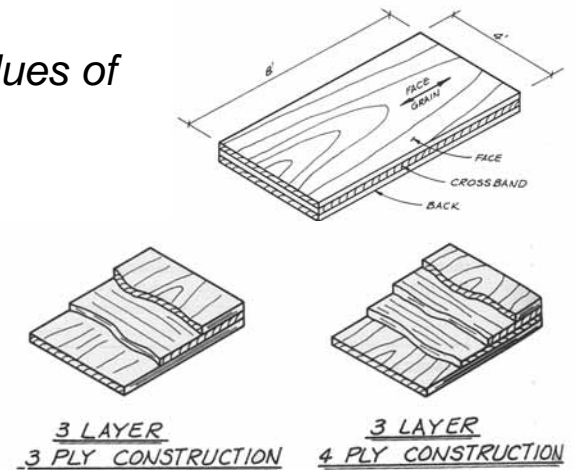
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Orthotropic Materials

- non-isometric
- directional values of E and μ
- ex:
 - plywood
 - laminates
 - polymer composites



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Stress Concentrations

- why we use f_{ave}
- increase in stress at changes in geometry
 - sharp notches
 - holes
 - corners

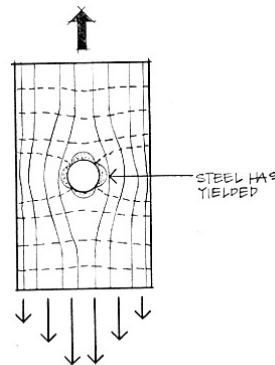
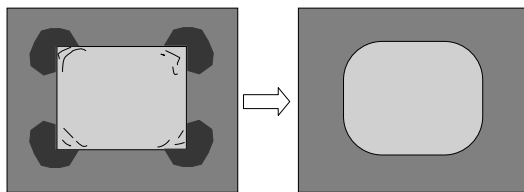


Figure 5.35 Stress trajectories around a hole.

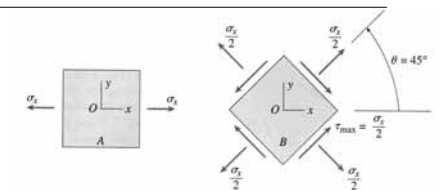
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Maximum Stresses

- if we need to know where max f and f_v happen:



$$\theta = 0^\circ \rightarrow \cos \theta = 1 \quad f_{\max} = \frac{P}{A_o}$$

$$\theta = 45^\circ \rightarrow \cos \theta = \sin \theta = \sqrt{0.5}$$

$$f_{v-\max} = \frac{P}{2A_o} = \frac{f_{\max}}{2}$$

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Maximum Stresses



FIG. 2-37 Shear failure along a 45° plane of a wood block loaded in compression



FIG. 2-38 Slip bands (or Lüders' bands) in a polished steel specimen loaded in tension

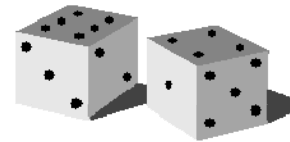
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Design of Members

- beyond allowable stress...
- materials aren't uniform 100% of the time
 - ultimate strength or capacity to failure may be different and some strengths hard to test for
- RISK & UNCERTAINTY



$$f_u = \frac{P_u}{A}$$

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Factor of Safety

- accommodate uncertainty with a safety factor:

$$\text{allowable load} = \frac{\text{ultimate load}}{F.S}$$

- with linear relation between load and stress:

$$F.S = \frac{\text{ultimate load}}{\text{allowable load}} = \frac{\text{ultimate stress}}{\text{allowable stress}}$$

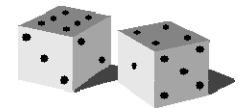
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Load and Resistance Factor Design

- loads on structures are
 - not constant
 - can be more influential on failure
 - happen more or less often
 - UNCERTAINTY



$$R_u = \gamma_D R_D + \gamma_L R_L \leq \phi R_n$$

ϕ - resistance factor

γ - load factor for (D)ead & (L)ive load

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