Architectural Structures I: Statics and Strength of Materials

ENDS 231 Dr. Anne Nichols Spring 2008

lecture Sixteen



elasticity & strain

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Deformations

- materials deform
- axially loaded materials change length
- normal stress is load per unit area
- STRAIN:
 - change in length over length
 - UNITLESS





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Shearing Strain

- deformations with shear
- parallelogram
- change in angles
- stress:
- strain: γ
 unitless (radians)

 \mathcal{T}



$$\gamma = \frac{\delta_s}{L} = \tan \phi \cong \phi$$

Shearing Strain

- deformations with torsion
- twist
- change in angle of line
- stress: au
- strain: γ
 unitless (radians)



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 $ho\phi$

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Load and Deformation

- for stress, need P & A
- for strain, need δ & L
 - how?
 - TEST with load and measure
 - plot P/A vs. ε







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Material Behavior

- every material has its own response
 - 10,000 psi
 - -L = 10 in
 - Douglas Fir vs. steel?



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Behavior Types • ductile - "necking" • true stress A • engineering stress - (simplified)



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Stress to Strain

• important to us in $f - \varepsilon$ diagrams:

- straight section
- LINEAR-ELASTIC
- recovers shape
 (no permanent
 deformation)





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Hooke's Law

- straight line has constant slope
- Hooke's Law

 $f = E \cdot \varepsilon$



- Modulus of elasticity

Young's modulusunits just like stress



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Stiffness

• ability to resist strain



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Isotropy & Anisotropy

- ISOTROPIC
 - materials with E <u>same</u> at any direction of loading
 - ex. steel
- ANISOTROPIC
 - materials with <u>different</u> E at any direction of loading
 - ex. wood is <u>orthotropic</u>



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Elastic, Plastic, Fatigue

- elastic springs back
- plastic has permanent deformation
- fatigue caused by reversed loading cycles





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Plastic Behavior

• ductile





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Lateral Strain

 or "what happens to the cross section with axial stress"



 $f_{y} = f_{z} = 0$



E

- strain in lateral direction
 - negative
 - equal for isometric materials



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Poisson's Ratio

• constant relationship between longitudinal strain and lateral strain

 $\mu = -\frac{lateral strain}{axial strain} = -\frac{\varepsilon_{y}}{\varepsilon_{x}} = -\frac{\varepsilon_{z}}{\varepsilon_{x}}$ $\varepsilon_{y} = \varepsilon_{z} = -\frac{\mu f_{x}}{E}$



 $0 < \mu < 0.5$



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Orthotropic Materials

- non-isometric
- directional values of E and μ
- ex:
 - plywood
 - laminates
 - polymercomposites







BACK

FACE

CROSS BAND

FACE



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Stress Concentrations

- why we use f_{ave}
- increase in stress at changes in geometry
 - sharp notches
 - holes
 - corners







Figure 5.35 Stress trajectories around a hole.

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Maximum Stresses

 if we need to know where max f and f_v happen:



max

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 $\theta = 0^{\circ} \to \cos \theta = 1 \qquad f_{\text{max}} = \frac{1}{A_o}$ $\theta = 45^{\circ} \to \cos \theta = \sin \theta = \sqrt{0.5}$

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v-max

Maximum Stresses



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Design of Members

- beyond allowable stress...
- materials aren't uniform 100% of the time
 - ultimate strength or capacity to failure may be different and some strengths hard to test for
- RISK & UNCERTAINTY





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Factor of Safety

- accommodate uncertainty with a safety factor: allowable load = $\frac{\text{ultimate load}}{F.S}$
- with linear relation between load and stress: $F.S = \frac{ultimate load}{allowable load} = \frac{ultimate stress}{allowable stress}$

Load and Resistance Factor Design

- loads on structures are
 - not constant



- can be more influential on failure
- happen more or less often
- UNCERTAINTY

 $R_{u} = \gamma_{D}R_{D} + \gamma_{L}R_{L} \leq \phi R_{n}$

 ϕ - resistance factor γ - load factor for (D)ead & (L)ive load

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