

**beams:  
 bending and shear**



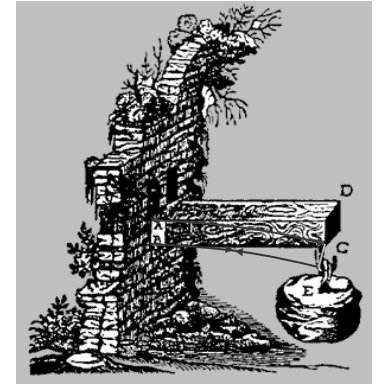
Beam Bending Stress 1  
 Lecture 18

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**Beam Bending**

- Galileo
  - relationship between stress and depth<sup>2</sup>
- can see
  - top squishing
  - bottom stretching
- what are the stress across the section?



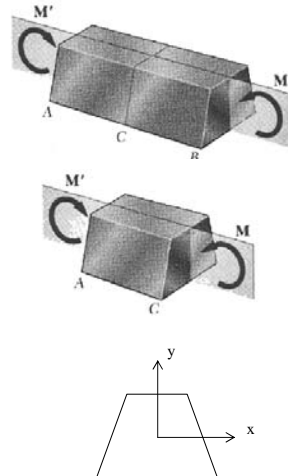
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**Pure Bending**

- bending only
- no shear
- axial normal stresses from bending can be found in
  - homogeneous materials
  - plane of symmetry
  - follow Hooke's law



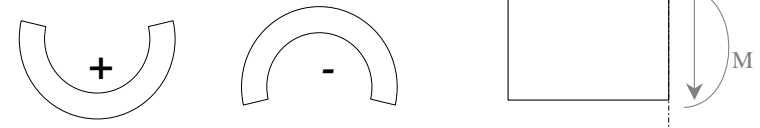
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**Bending Moments**

- sign convention:



- size of maximum internal moment will govern our design of the section

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# Normal Stresses

- **geometric fit**
  - plane sections remain plane
  - stress varies linearly

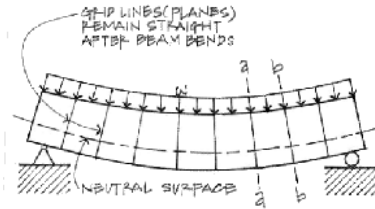


Figure 8.5(b) Beam bending under load.

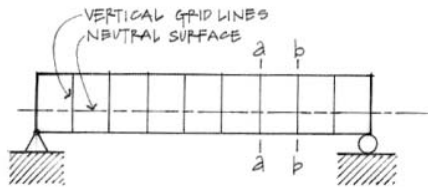
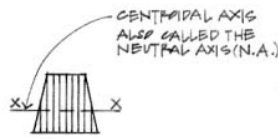


Figure 8.5(a) Beam elevation before loading.



Beam cross section.

# Neutral Axis

- stresses vary linearly
- zero stress occurs at the centroid
- neutral axis is line of centroids (n.a.)

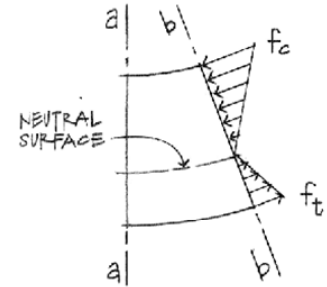


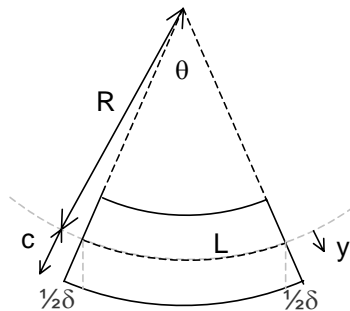
Figure 8.8 Bending stresses on section b-b.

# Derivation of Stress from Strain

- pure bending = arc shape

$$L = R\theta$$

$$L_{outside} = (R + y)\theta$$



$$\epsilon = \frac{\delta}{L} = \frac{L_{outside} - L}{L} = \frac{(R + y)\theta - R\theta}{R\theta} = \frac{y}{R}$$

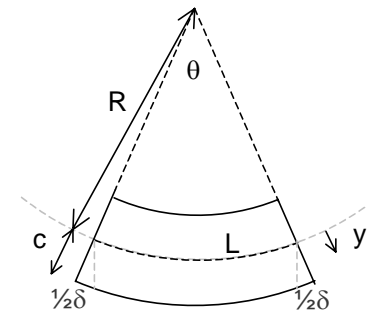
# Derivation of Stress

- zero stress at n.a.

$$f = E\epsilon = \frac{Ey}{R}$$

$$f_{max} = \frac{Ec}{R}$$

$$f = \frac{y}{c} f_{max}$$



## Bending Moment

- resultant moment from stresses = bending moment!

$$M = \Sigma fy\Delta A$$

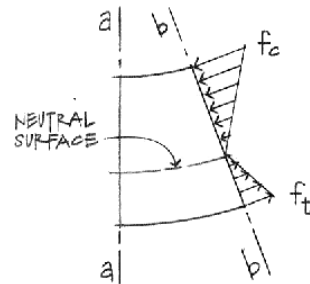


Figure 8.8 Bending stresses on section b-b.

$$= \Sigma \frac{yf_{max}}{c} y\Delta A = \frac{f_{max}}{c} \Sigma y^2 \Delta A = \frac{f_{max}}{c} I = f_{max} S$$

## Bending Stress Relations

$$\frac{1}{R} = \frac{M}{EI}$$

curvature

$$f_b = \frac{My}{I}$$

general bending stress

$$S = \frac{I}{c}$$

section modulus

$$f_b = \frac{M}{S}$$

maximum bending stress

$$S_{required} \geq \frac{M}{F_b}$$

required section modulus for design