ARCHITECTURAL STRUCTURES I: STATICS AND STRENGTH OF MATERIALS

ENDS 231

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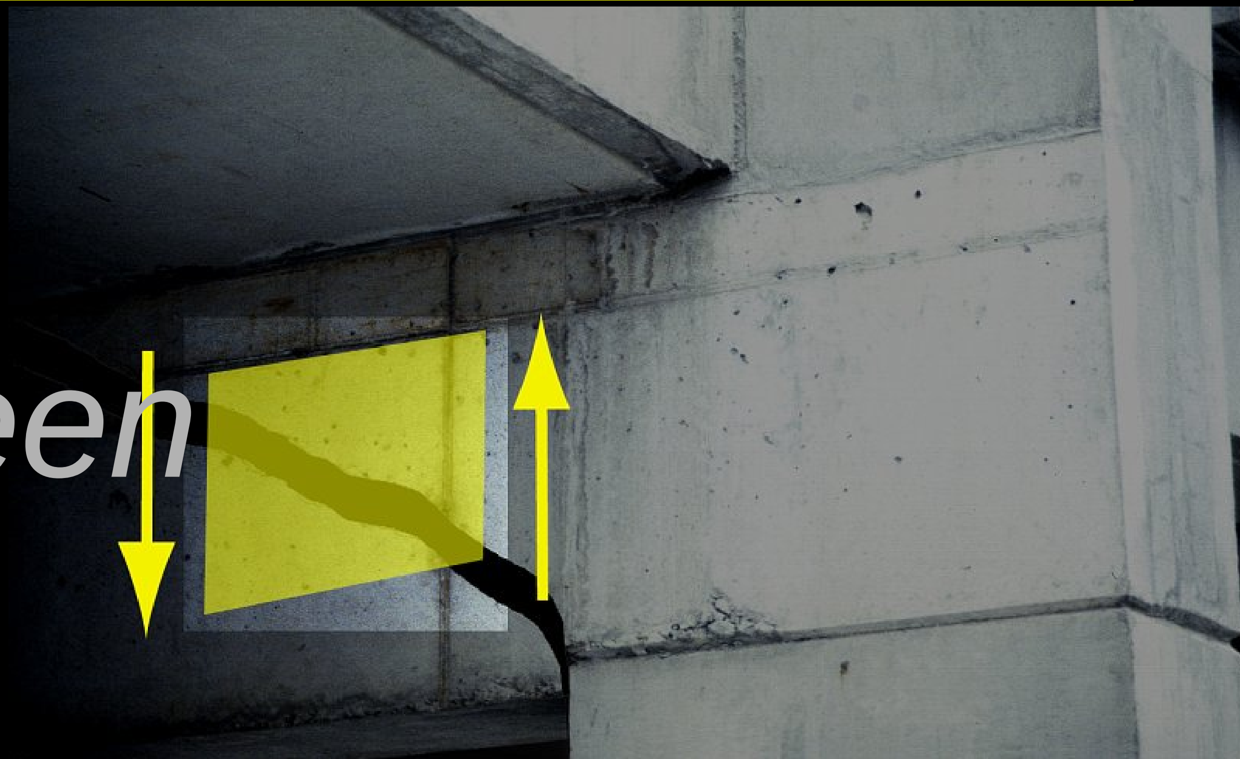
SUMMER 2006

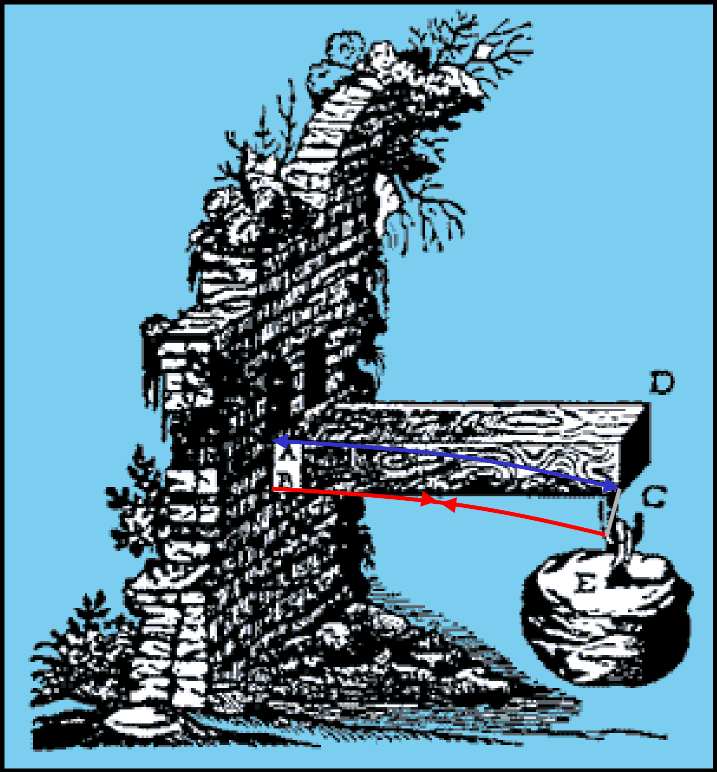
lecture
seventeen



beams: bending and shear

Beam Bending

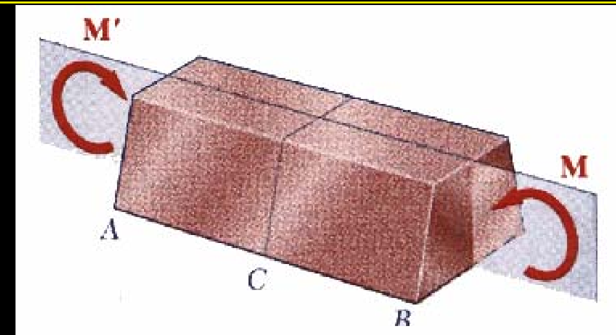
- Galileo
  - relationship between stress and depth^2
- can see
  - top squishing
  - bottom stretching

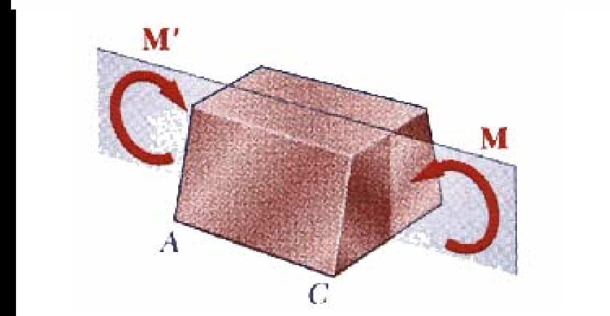


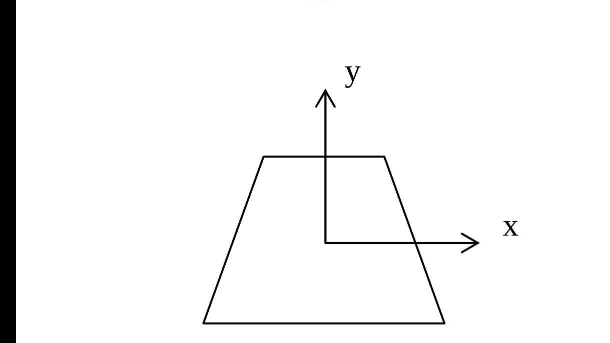
- what are the stress across the section?

Pure Bending

- *bending only*
- *no shear*
- *axial normal stresses from bending can be found in*
  - *homogeneous materials*
  - *plane of symmetry*
  - *follow Hooke's law*

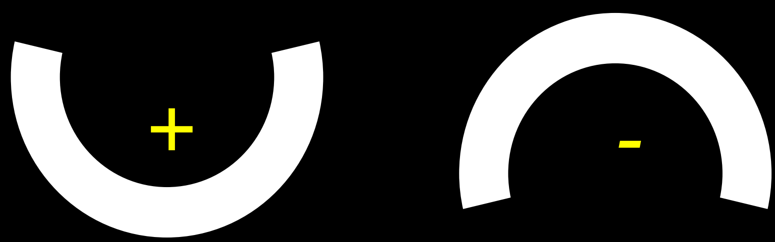


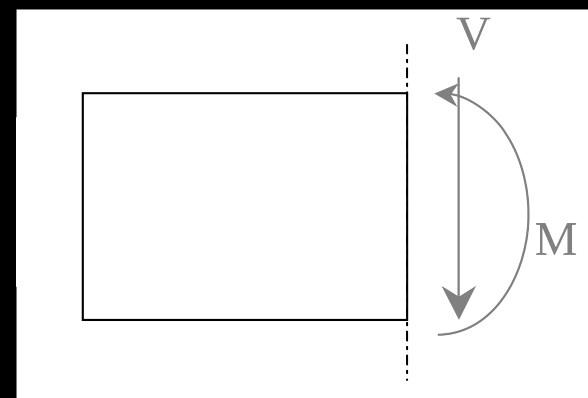




Bending Moments

- sign convention:





- size of maximum internal moment will govern our design of the section

Normal Stresses

- geometric fit
  - plane sections remain plane
  - stress varies linearly

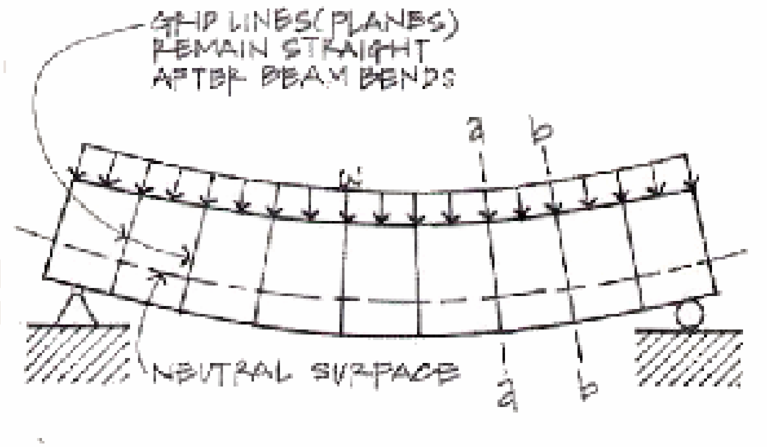


Figure 8.5(b) Beam bending under load.

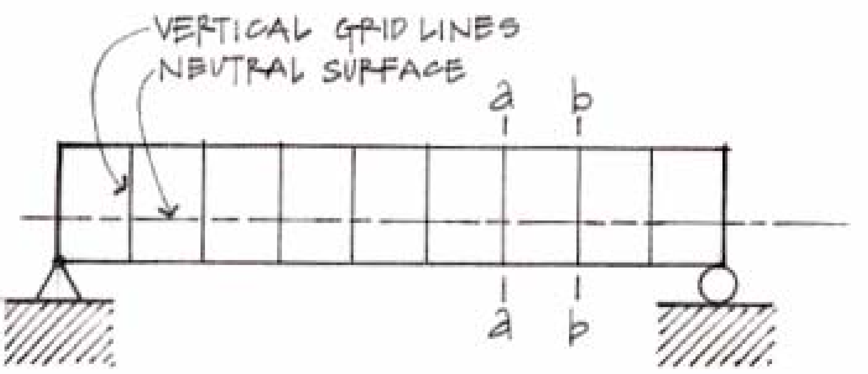
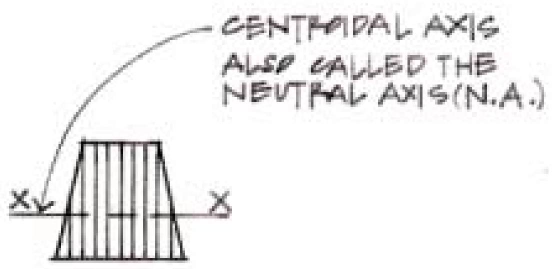


Figure 8.5(a) Beam elevation before loading.



Beam cross section.

Neutral Axis

- *stresses vary linearly*
- *zero stress occurs at the centroid*
- *neutral axis is line of centroids (n.a.)*

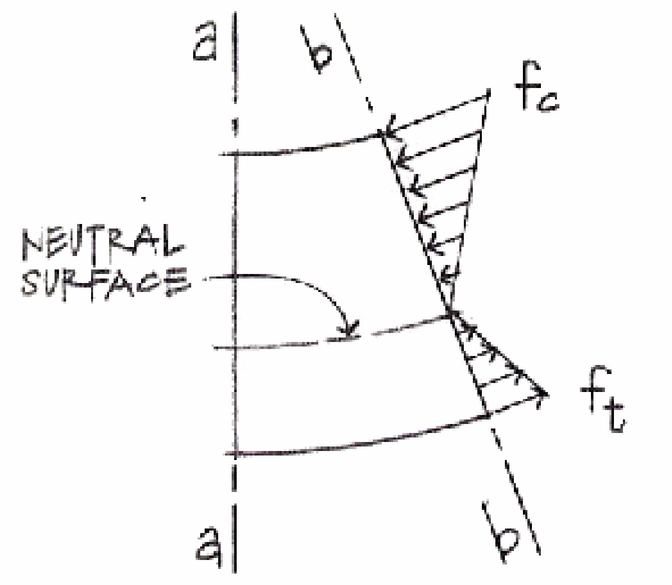


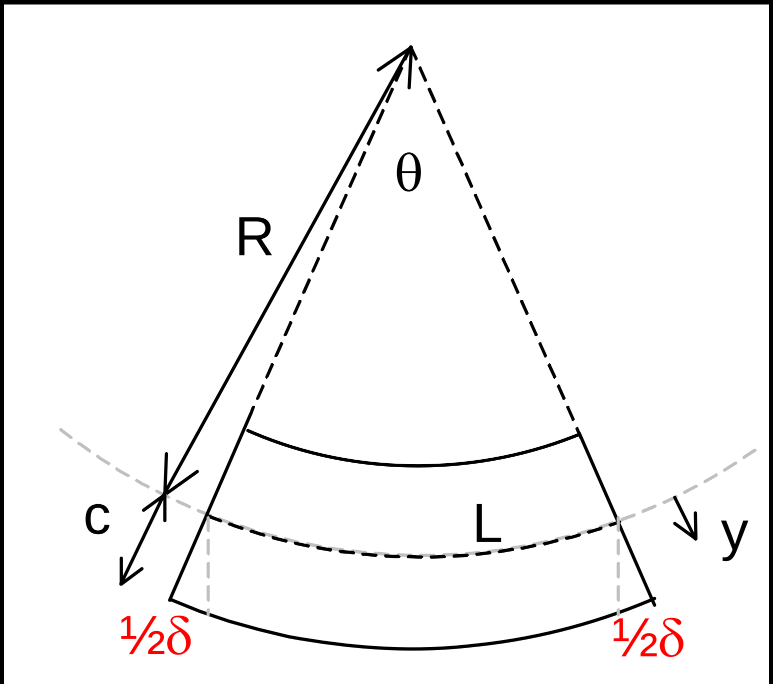
Figure 8.8 Bending stresses on section b-b.

Derivation of Stress from Strain

- pure bending = arc shape

$$L = R\theta$$

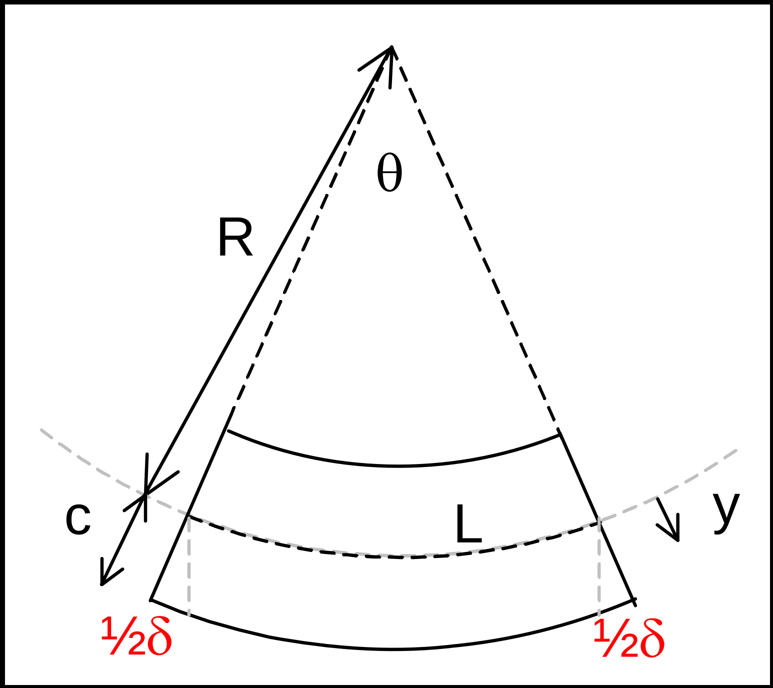
$$L_{outside} = (R + y)\theta$$



$$\varepsilon = \frac{\delta}{L} = \frac{L_{outside} - L}{L} = \frac{(R+y)\theta - R\theta}{R\theta} = \frac{y}{R}$$

Derivation of Stress

- zero stress at n.a.



$$f = E\varepsilon = \frac{Ey}{R}$$

$$f_{\max} = \frac{Ec}{R}$$

$$f = \frac{y}{c} f_{\max}$$

Bending Moment

- *resultant moment from stresses = bending moment!*

$$M = \Sigma f y \Delta A$$

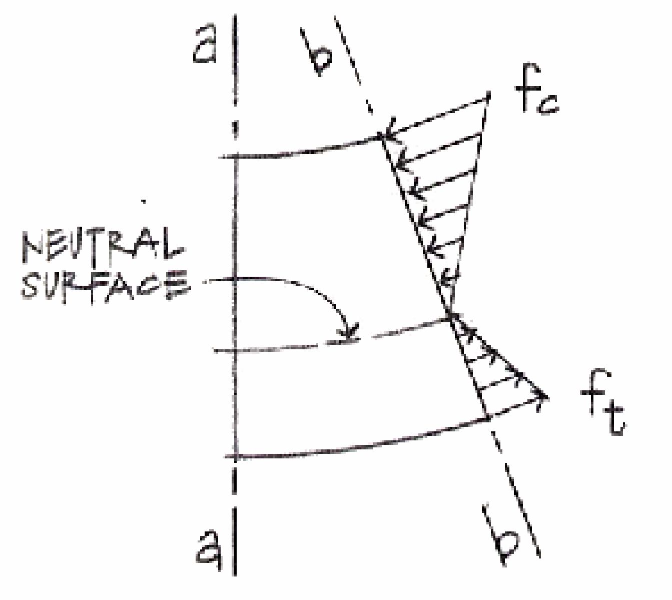


Figure 8.8 Bending stresses on section b-b.

$$= \Sigma \frac{y f_{max}}{c} y \Delta A = \frac{f_{max}}{c} \underline{\Sigma y^2 \Delta A} = \frac{f_{max}}{c} I = f_{max} S$$

Bending Stress Relations

$$\frac{1}{R} = \frac{M}{EI}$$

curvature

$$f_b = \frac{My}{I}$$

general bending stress

$$S = \frac{I}{c}$$

section modulus

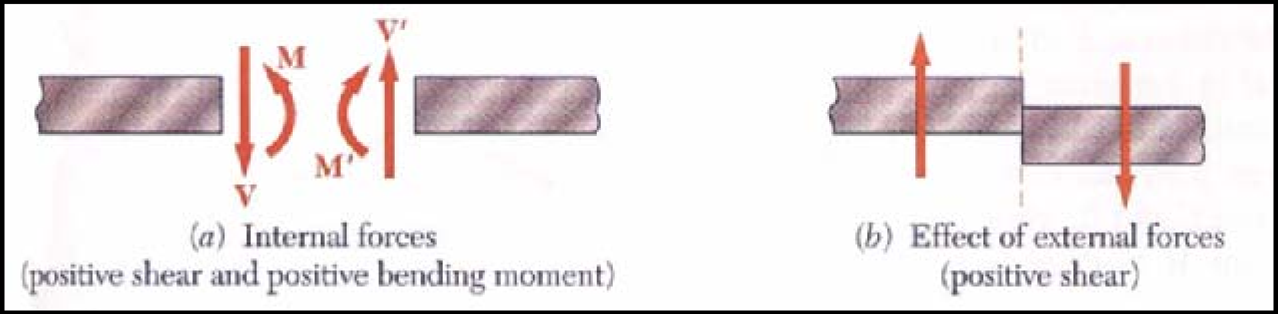
$$f_b = \frac{M}{S}$$

maximum bending stress

$$S_{required} \geq \frac{M}{F_b}$$

required section modulus for design

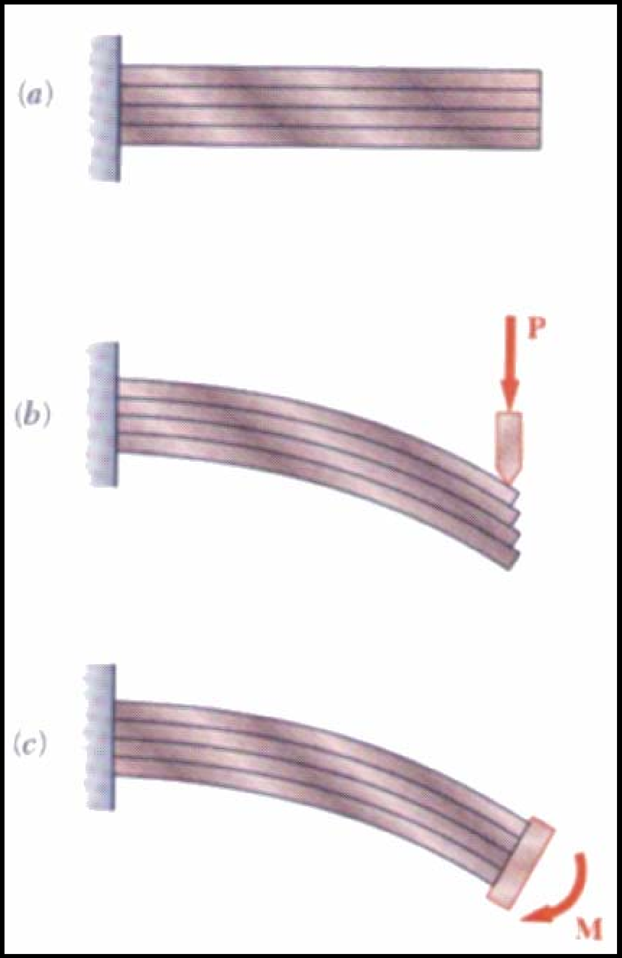
Transverse Loading and Shear



- *perpendicular loading*
- *internal shear*
- *along with bending moment*

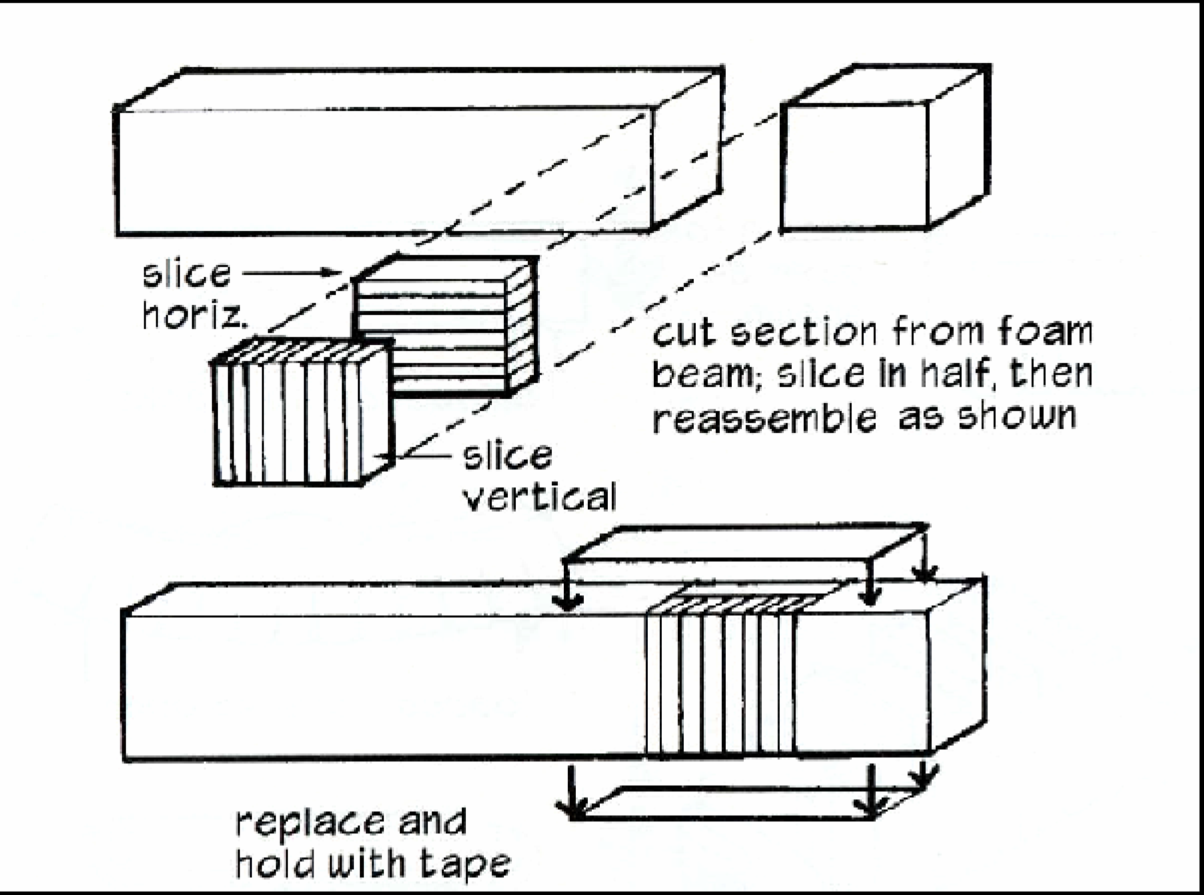
Bending vs. Shear in Design

- *bending stresses dominate*
- *shear stresses exist horizontally with shear*
- *<u>no shear stresses</u> with pure bending*



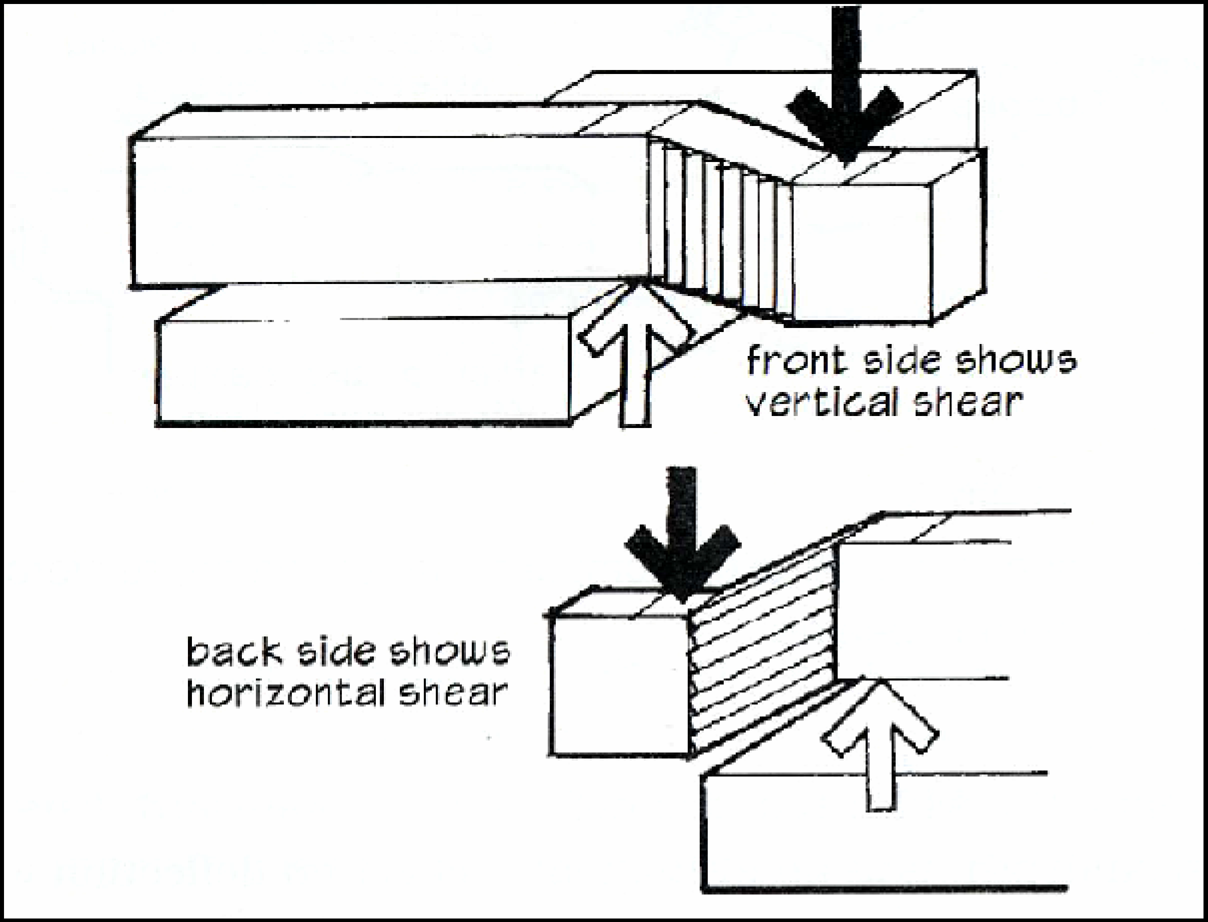
Shear Stresses

- *horizontal & vertical*



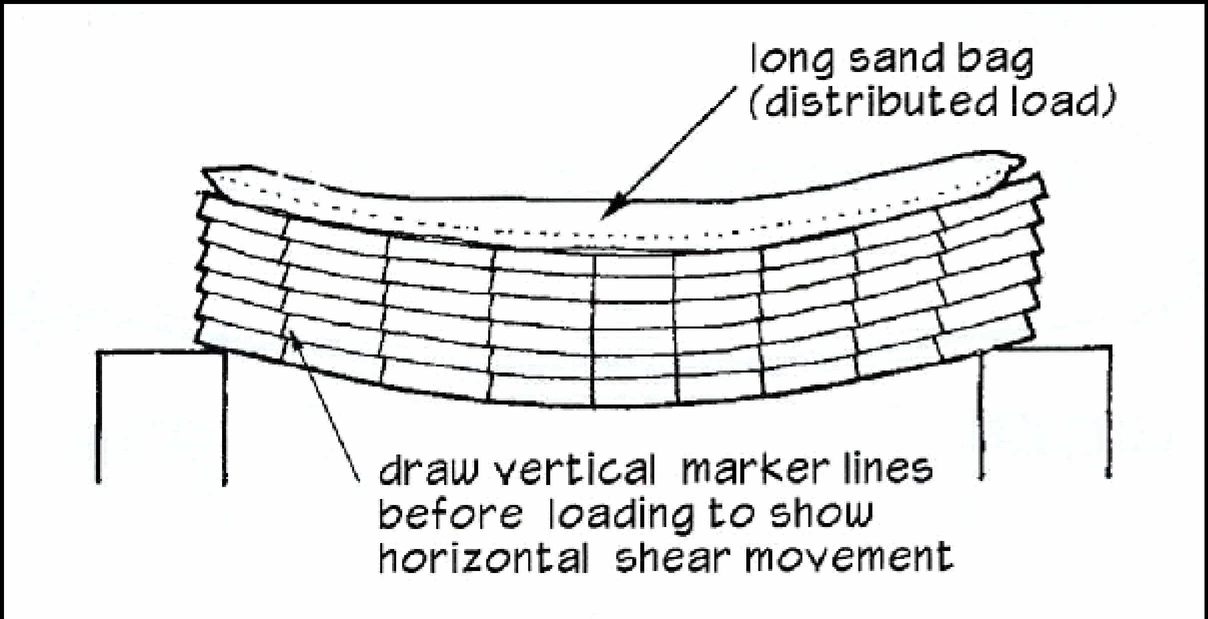
Shear Stresses

- *horizontal & vertical*



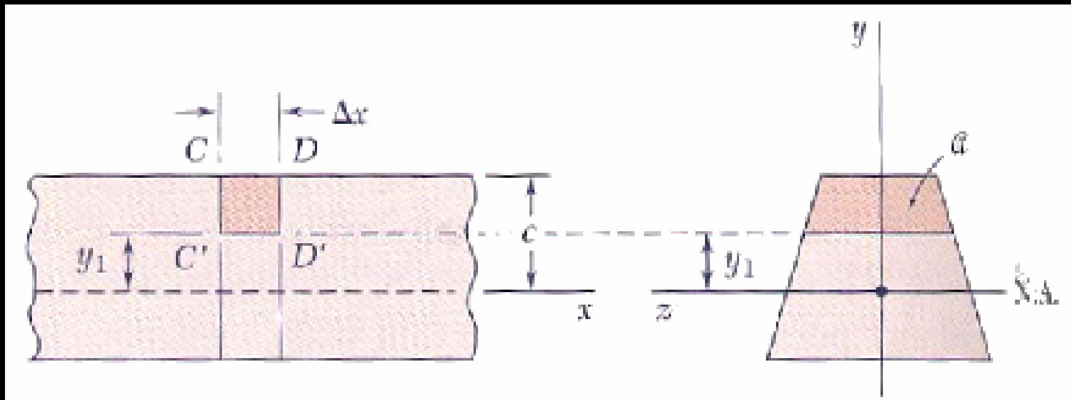
Beam Stresses

- *horizontal with bending*

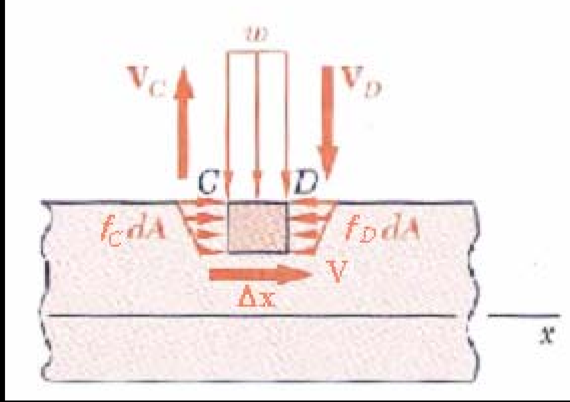


Equilibrium

- horizontal force V needed



$$V_{longitudinal} = \frac{V_T Q}{I} \Delta x$$

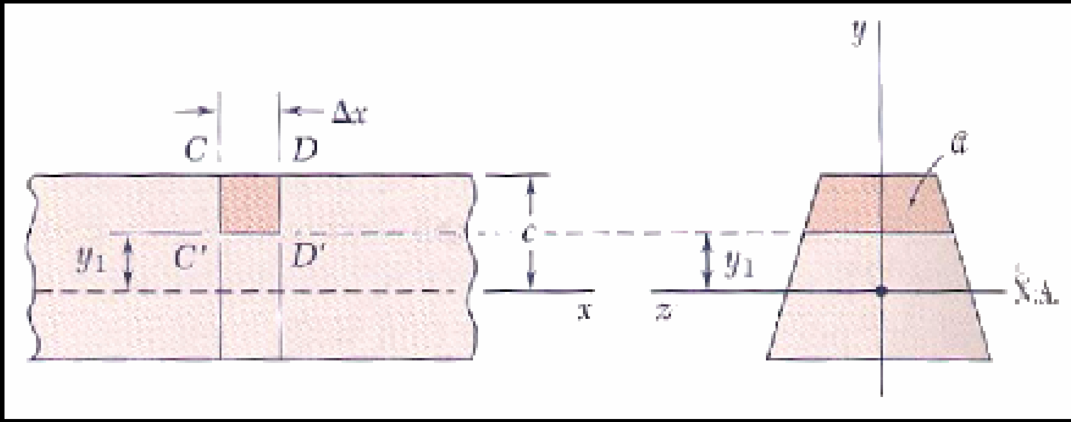


- Q is a moment area

Moment of Area

- *Q is a moment area with respect to the n.a. of area above or below the horizontal*

- *Q_{max} at y=0 (n.a.)*



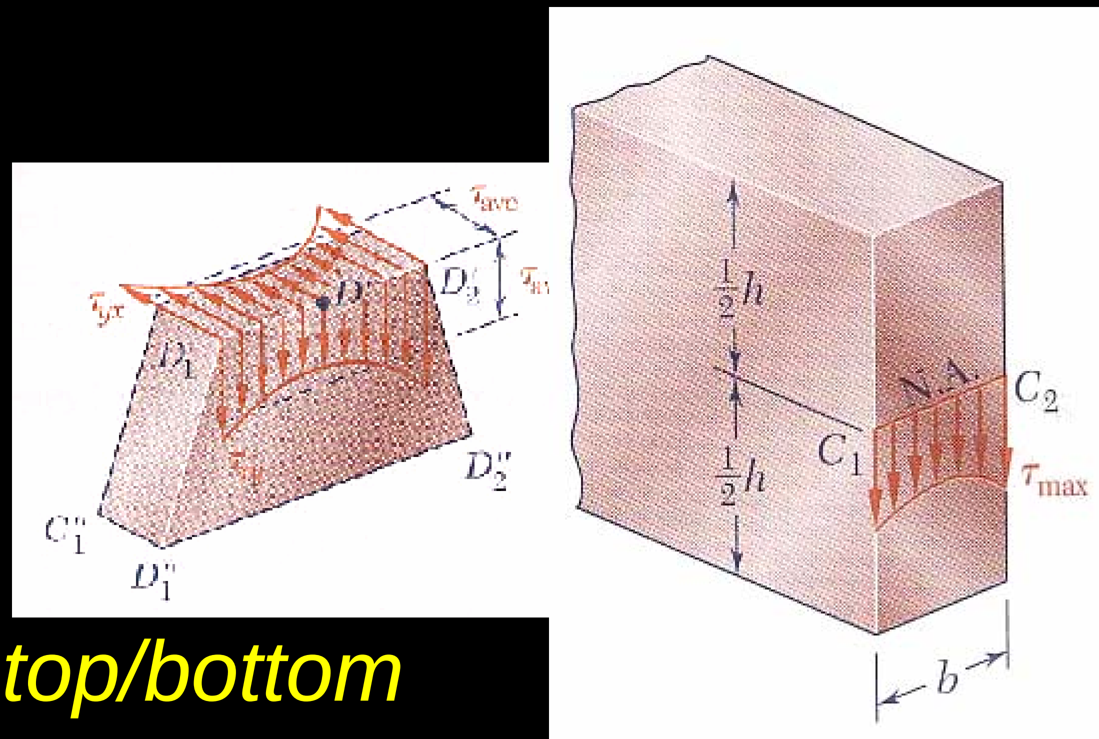
- *q is shear flow:*

$$q = \frac{V_{longitudinal}}{\Delta x} = \frac{V_T Q}{I}$$

Shearing Stresses

$$f_v = \frac{V}{\Delta A} = \frac{V}{b \cdot \Delta x}$$

$$f_{v-ave} = \frac{VQ}{Ib}$$



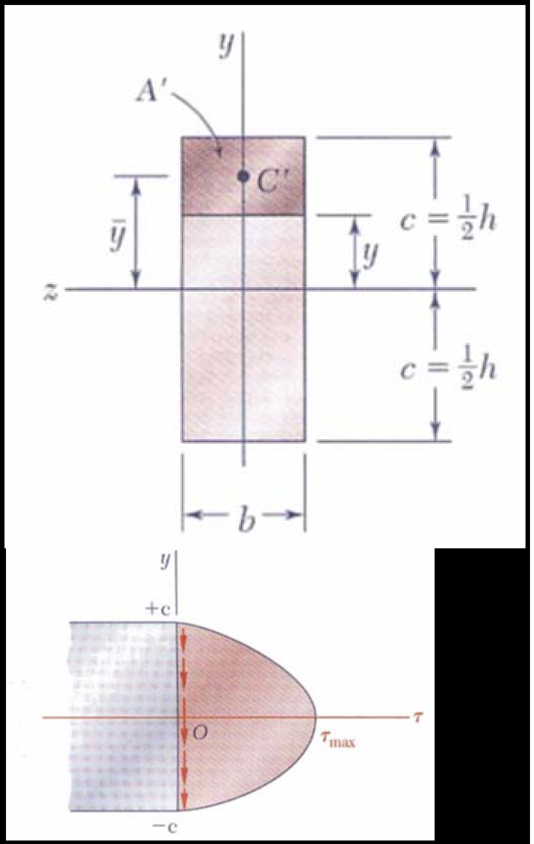
- f_{v-ave} = 0 on the top/bottom
- *b min may not be with Q max*
- *with h/4* $\geq b$, $f_{v\text{-}max} \leq 1.008\, f_{v\text{-}ave}$

Rectangular Sections

$$I = \frac{bh^3}{12} \qquad Q = A\bar{y} = {bh^2}/{8}$$

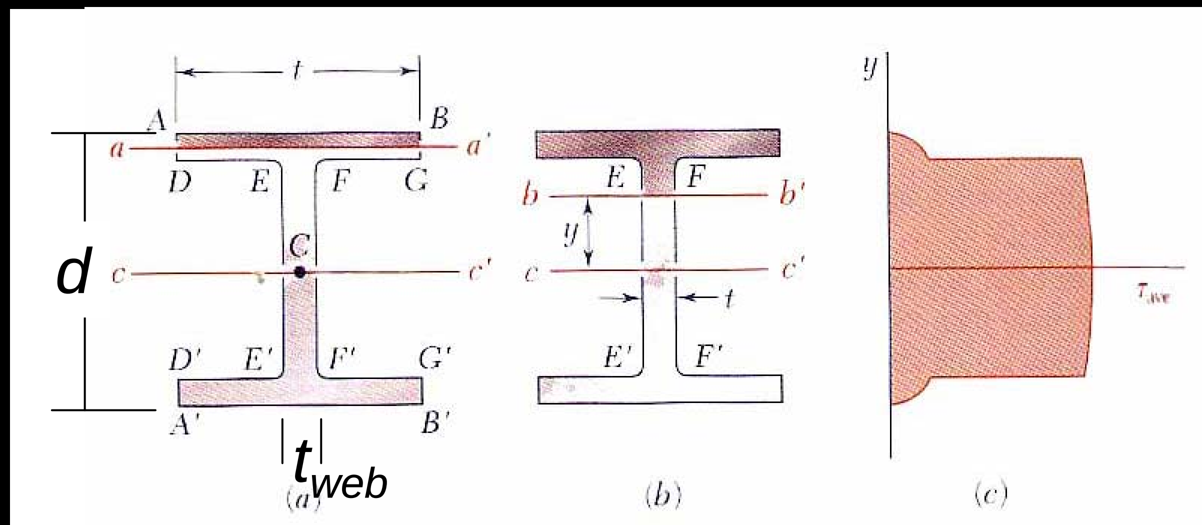
$$f_v = \frac{VQ}{Ib} = \frac{3V}{2A}$$

- $f_{v\text{-}max}$ occurs at n.a.



Steel Beam Webs

- W and S sections
  - b varies



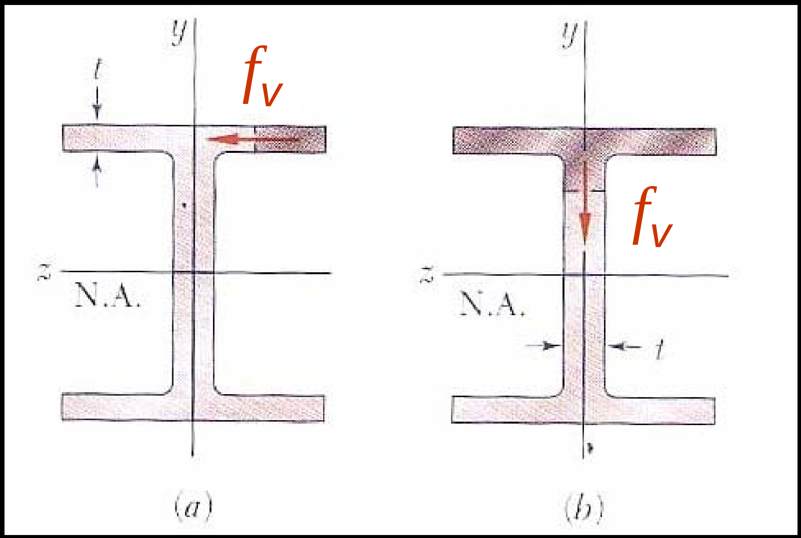
  - stress in flange negligible
  - presume constant stress in web

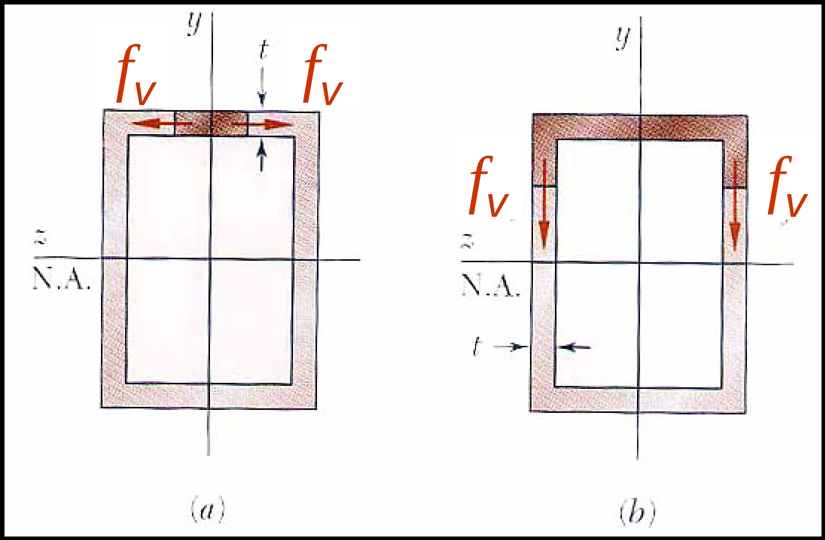
$$f_{v-\max} = \frac{3V}{2A} \approx \frac{V}{A_{web}}$$

Shear Flow

- *loads applied in plane of symmetry*
- *cut made perpendicular*

$$q = \frac{VQ}{I}$$

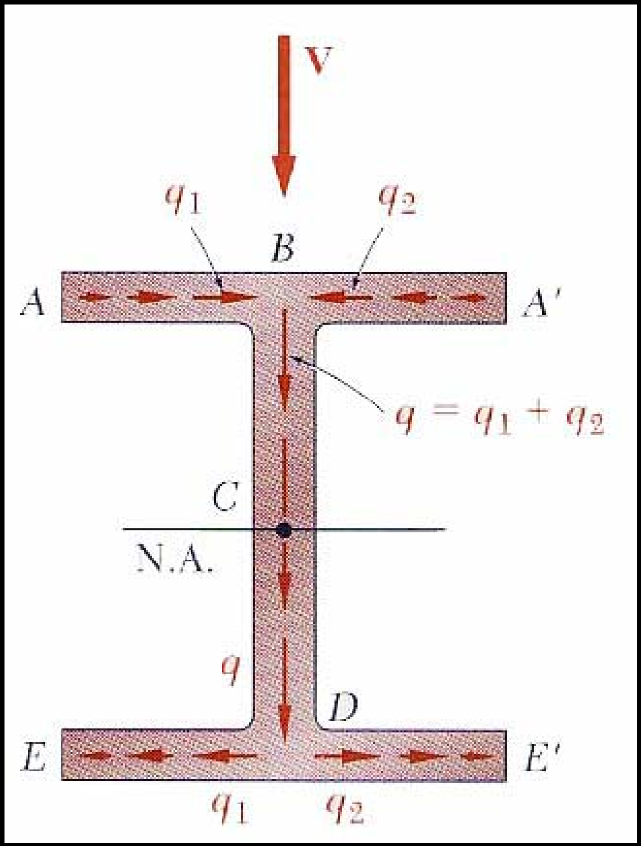




Shear Flow Quantity

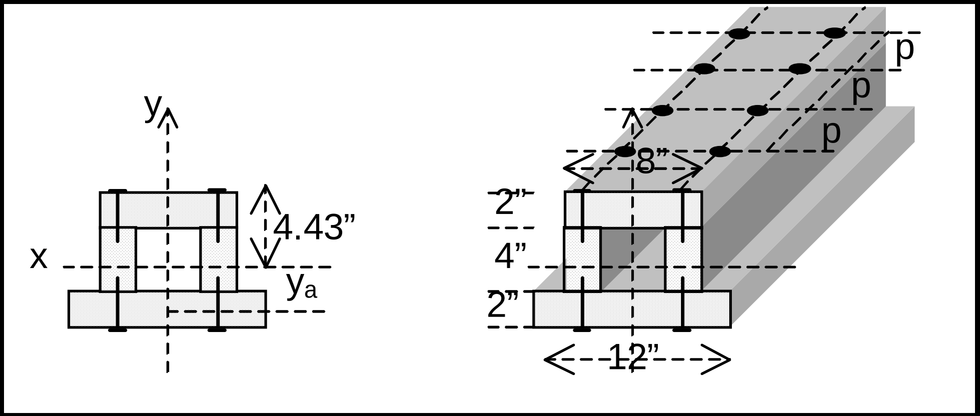
- sketch from Q

$$q = \frac{VQ}{I}$$



Connectors Resisting Shear

- plates with
  - nails
  - rivets
  - bolts
- splices



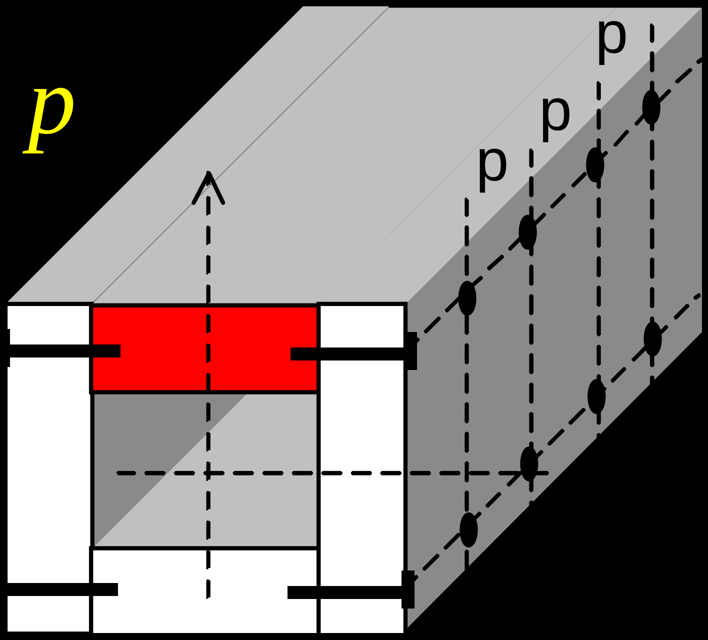
$$\frac{V_{longitudinal}}{p} = \frac{VQ}{I}$$

$$nF_{connector} \geq \frac{VQ_{connected\ area}}{I} \cdot p$$

Vertical Connectors

- *isolate an area with vertical interfaces*

$$nF_{connector} \geq \frac{VQ_{connected\ area}}{I} \cdot p$$



Unsymmetrical Shear or Section

- member can bend and twist
  - not symmetric
  - shear not in that plane
- shear center
  - moments balance