

**ARCHITECTURAL STRUCTURES I:  
STATICS AND STRENGTH OF MATERIALS**

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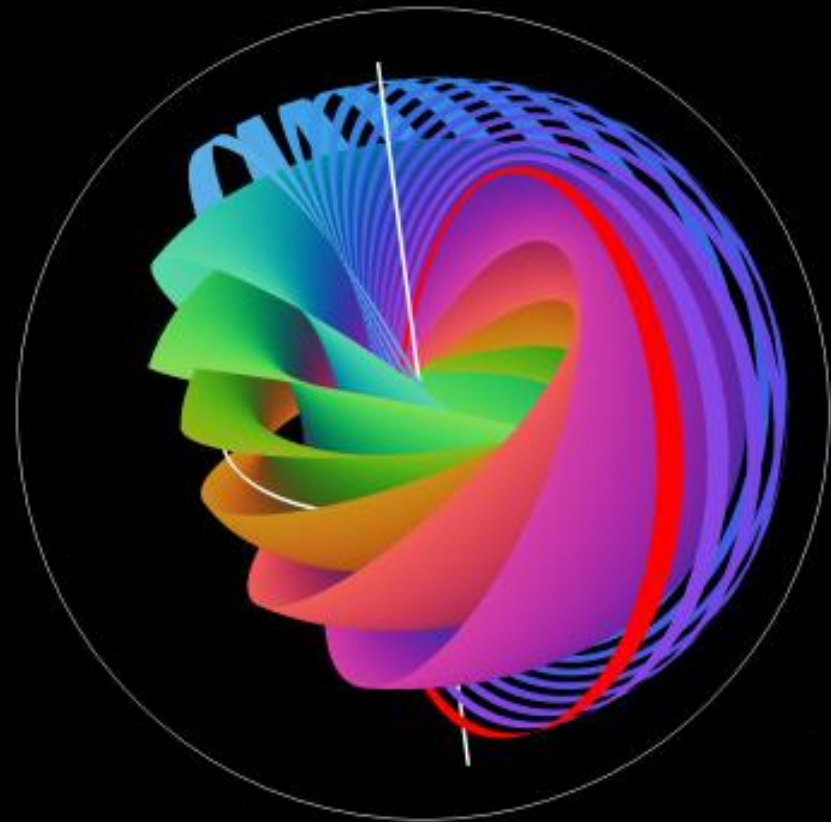
**ENDS 231**

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**SUMMER 2006**

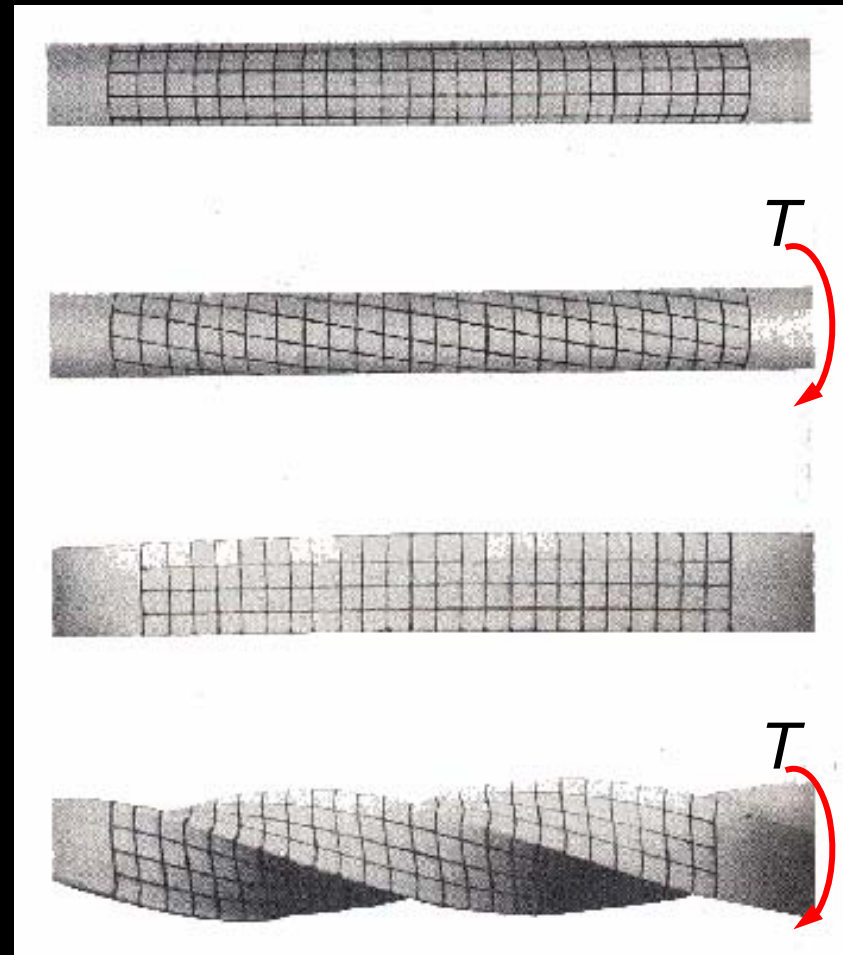
**lecture  
sixteen**

**torsion  
& thermal effects**



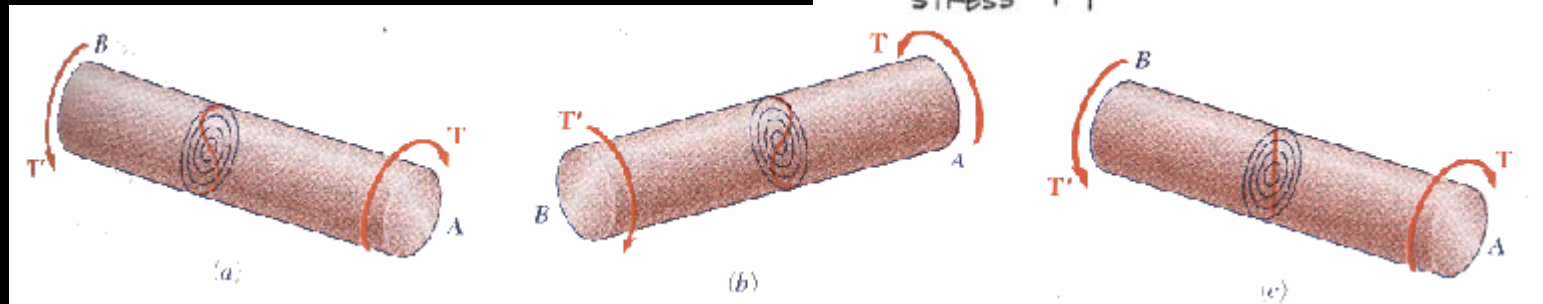
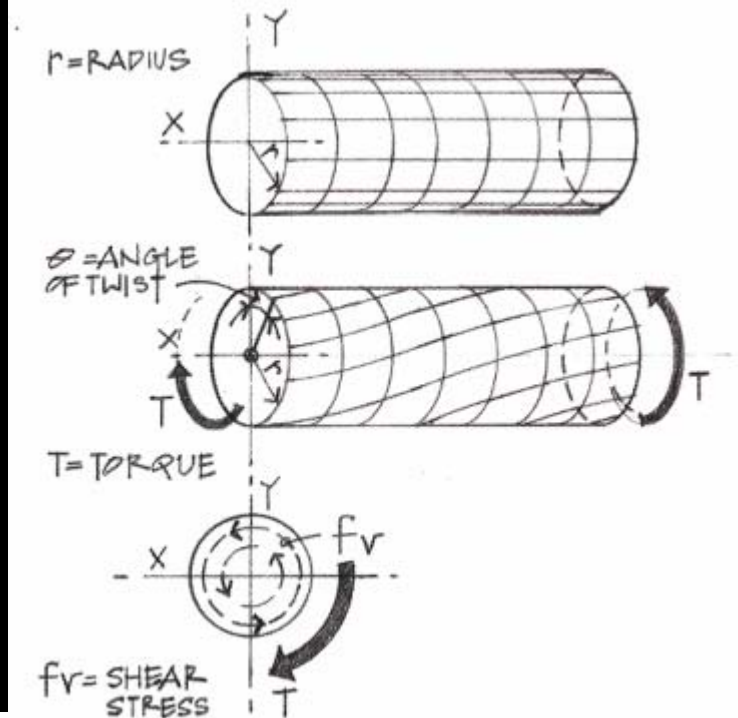
# Torsional Stress & Strain

- *can see torsional stresses & twisting of axi-symmetrical cross sections*
  - torque
  - *remain plane*
  - *undistorted*
  - *rotates*
- *not true for square sections....*



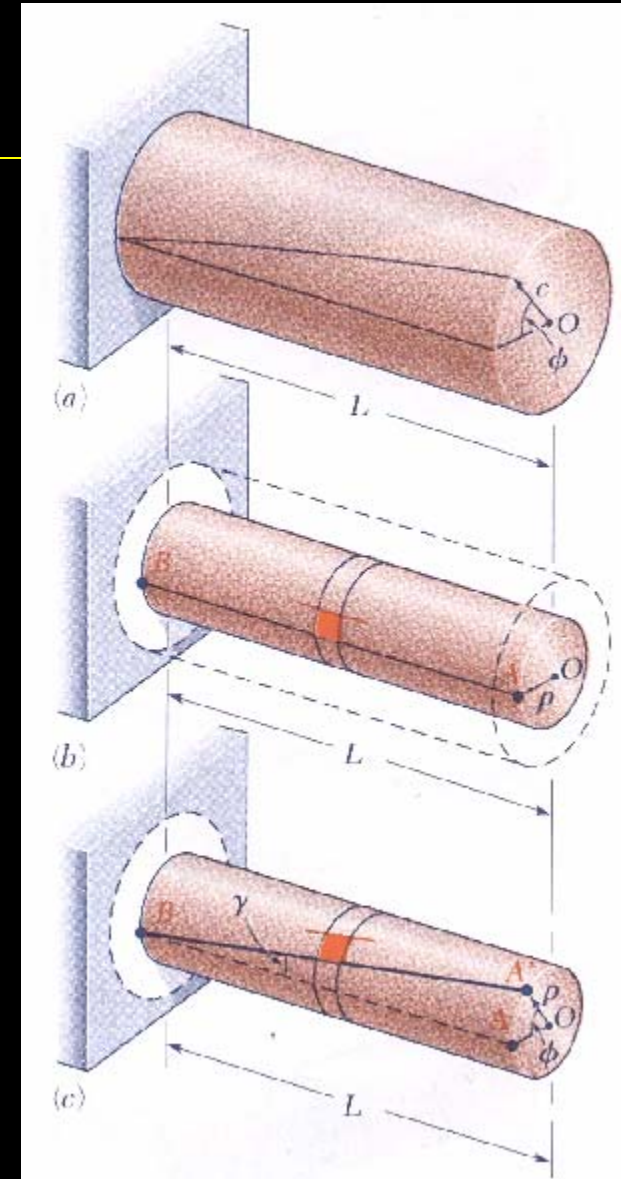
# Shear Stress Distribution

- *depend on the deformation*
- $\phi = \text{angle of twist}$   
– *measure*
- *can prove planar section doesn't distort*



# Shearing Strain

- related to  $\phi$  
$$\gamma = \frac{\rho\phi}{L}$$
- $\rho$  is the radial distance from the centroid to the point under strain
- shear strain varies linearly along the radius:  $\gamma_{max}$  is at outer diameter



# *Torsional Stress - Strain*

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- *know  $f_v = \tau = G \cdot \gamma$  and  $\gamma = \frac{\rho\phi}{L}$*
- *so  $\tau = G \cdot \frac{\rho\phi}{L}$*
- *where  $G$  is the Shear Modulus*

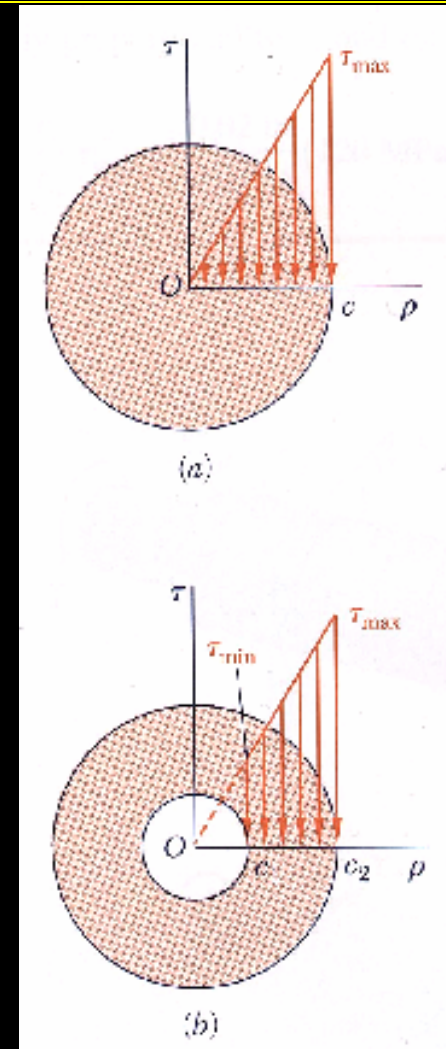
# Torsional Stress - Strain

- from  $T = \Sigma \tau(\rho) \Delta A$

- can derive  $T = \frac{\tau J}{\rho}$

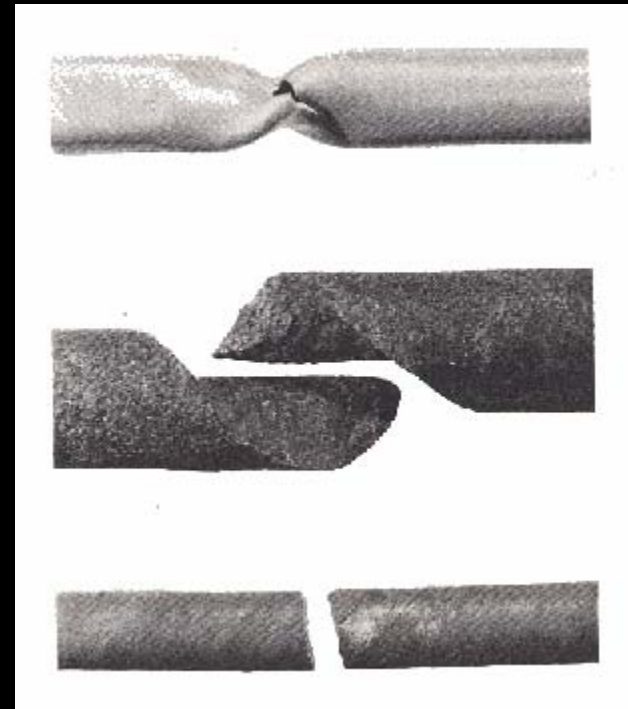
- where  $J$  is the polar moment of inertia

- elastic range  $\tau = \frac{T\rho}{J}$



# Shear Stress

- $\tau_{max}$  happens at outer diameter
- *combined shear and axial stresses*
  - *maximum shear stress at 45° “twisted” plane*



# Shear strain

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- knowing  $\tau = G \cdot \frac{\rho\phi}{L}$  and  $\tau = \frac{T\rho}{J}$

- solve:  $\phi = \frac{TL}{JG}$

- composite shafts:  $\phi = \sum_i \frac{T_i L_i}{J_i G_i}$

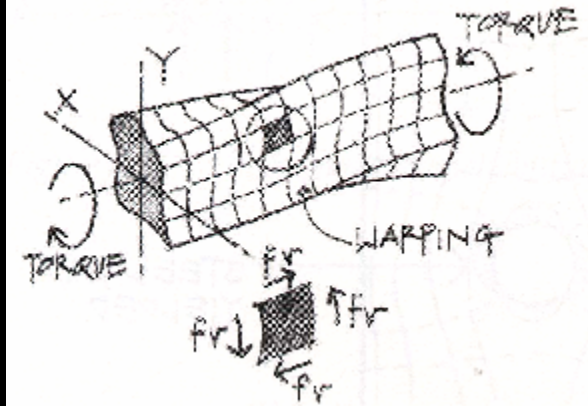


# Noncircular Shapes

- torsion depends on  $J$
- plane sections don't remain plane
- $\tau_{\max}$  is still at outer diameter

$$\tau_{\max} = \frac{T}{c_1 ab^2} \quad \phi = \frac{TL}{c_2 ab^3 G}$$

– where  $a$  is longer side ( $> b$ )

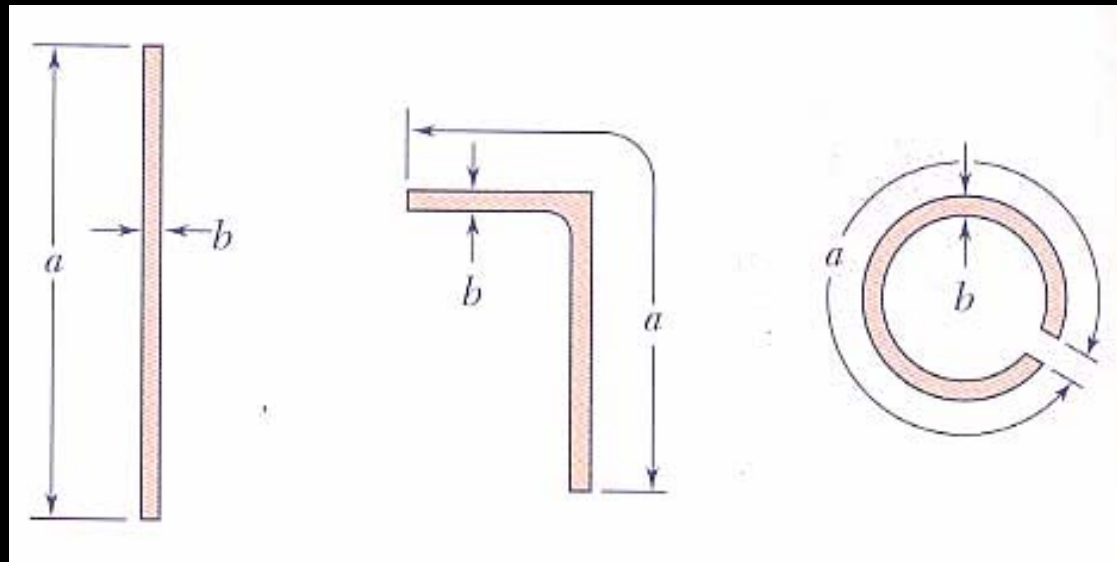


**TABLE 3.1. Coefficients for Rectangular Bars in Torsion**

$a/b$	$c_1$	$c_2$
1.0	0.208	0.1406
1.2	0.219	0.1661
1.5	0.231	0.1958
2.0	0.246	0.229
2.5	0.258	0.249
3.0	0.267	0.263
4.0	0.282	0.281
5.0	0.291	0.291
10.0	0.312	0.312
$\infty$	0.333	0.333

# Open Thin-Walled Sections

- with very large  $a/b$  ratios:



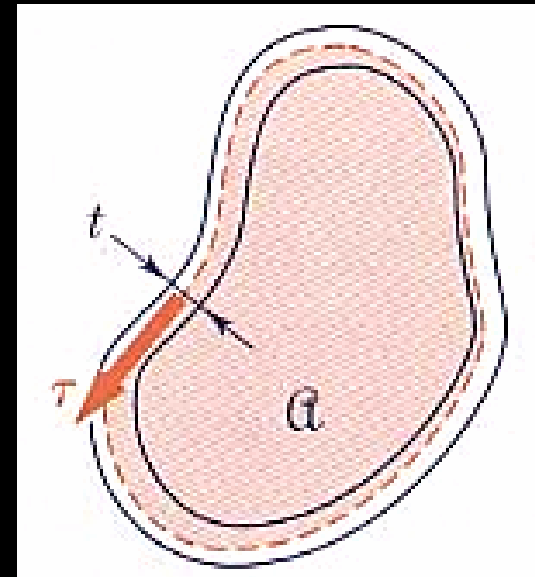
$$\tau_{\max} = \frac{T}{\frac{1}{3} ab^2} \quad \phi = \frac{TL}{\frac{1}{3} ab^3 G}$$

# Shear Flow in Closed Sections

- $q$  is the internal shear force/unit length

$$\tau = \frac{T}{2t\mathcal{A}}$$

$$\phi = \frac{TL}{4t\mathcal{A}^2} \sum_i \frac{s_i}{t_i}$$



- $\mathcal{A}$  is the area bounded by the centerline
- $s_i$  is the length segment,  $t_i$  is the thickness

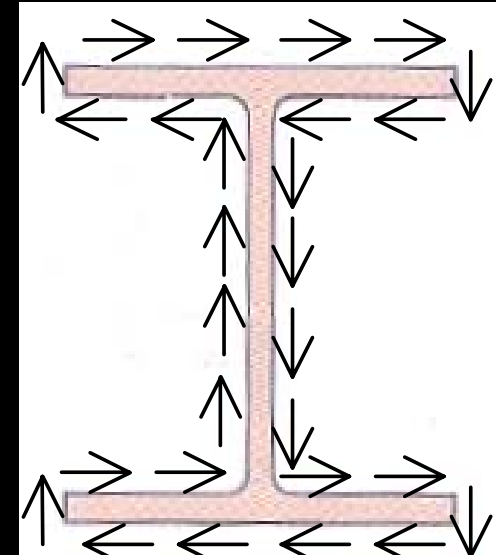
# Shear Flow in Open Sections

- each segment has proportion of  $T$  with respect to torsional rigidity,

$$\tau_{\max} = \frac{T t_{\max}}{\frac{1}{3} \sum b_i t_i^3}$$

- total angle of twist:

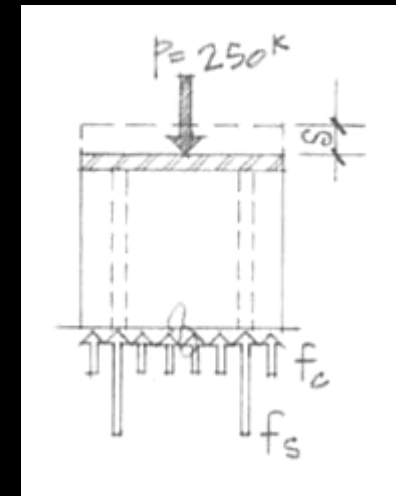
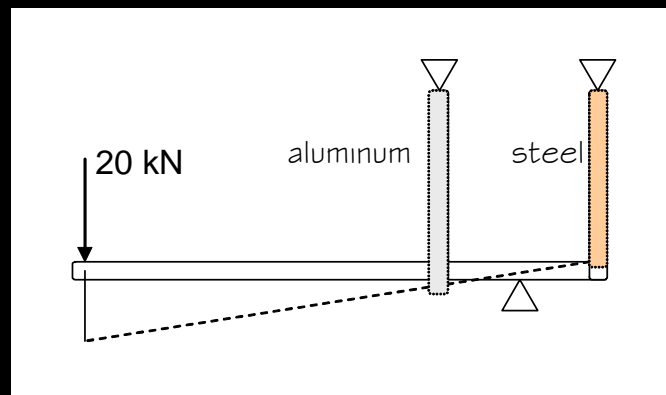
$$\phi = \frac{TL}{\frac{1}{3} G \sum b_i t_i^3}$$



- *I* beams - web is thicker, so  $\tau_{\max}$  is in web

# Deformation Relationships

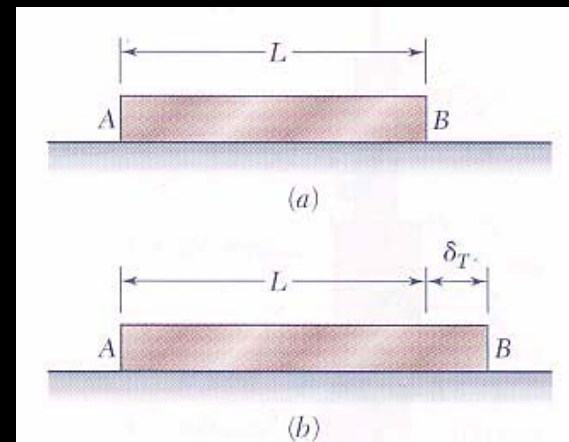
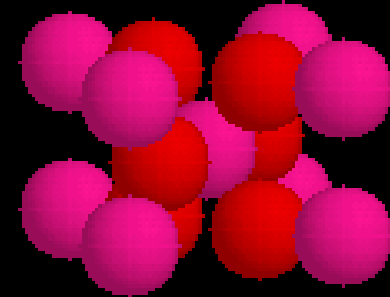
- *physical movement*
  - axially (same or zero)
  - rotations from axial changes



- $\delta = \frac{PL}{AE}$  relates  $\delta$  to  $P$

# Deformations from Temperature

- *atomic chemistry reacts to changes in energy*
- *solid materials*
  - *can contract with decrease in temperature*
  - *can expand with increase in temperature*
- *linear change can be measured per degree*



# Thermal Deformation

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- $\alpha$  - the rate of strain per degree

- UNITS :  $/^{\circ}F$  ,  $/^{\circ}C$

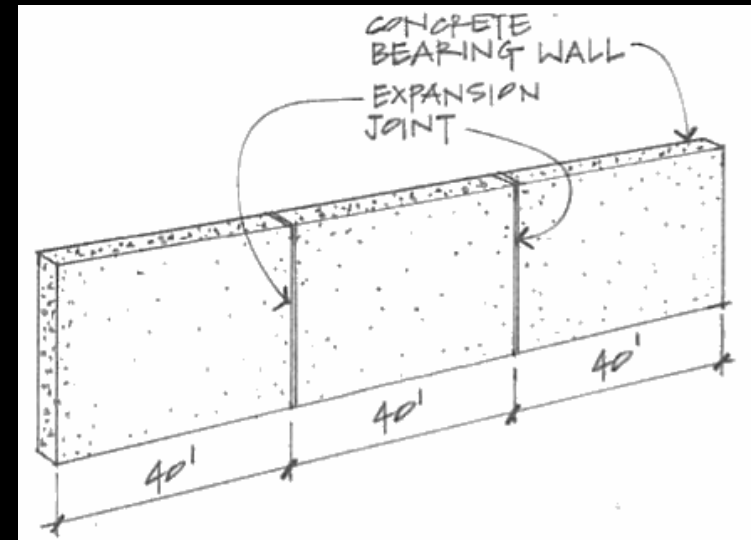
- length change:  $\delta_T = \alpha(\Delta T)L$

- thermal strain:  $\varepsilon_T = \alpha(\Delta T)$

– no stress when movement allowed

# Coefficients of Thermal Expansion

Material	Coefficients ( $\alpha$ ) [in./in./°F]
Wood	$3.0 \times 10^{-6}$
Glass	$4.4 \times 10^{-6}$
Concrete	$5.5 \times 10^{-6}$
Cast Iron	$5.9 \times 10^{-6}$
Steel	$6.5 \times 10^{-6}$
Wrought Iron	$6.7 \times 10^{-6}$
Copper	$9.3 \times 10^{-6}$
Bronze	$10.1 \times 10^{-6}$
Brass	$10.4 \times 10^{-6}$
Aluminum	$12.8 \times 10^{-6}$





# Stresses and Thermal Strains

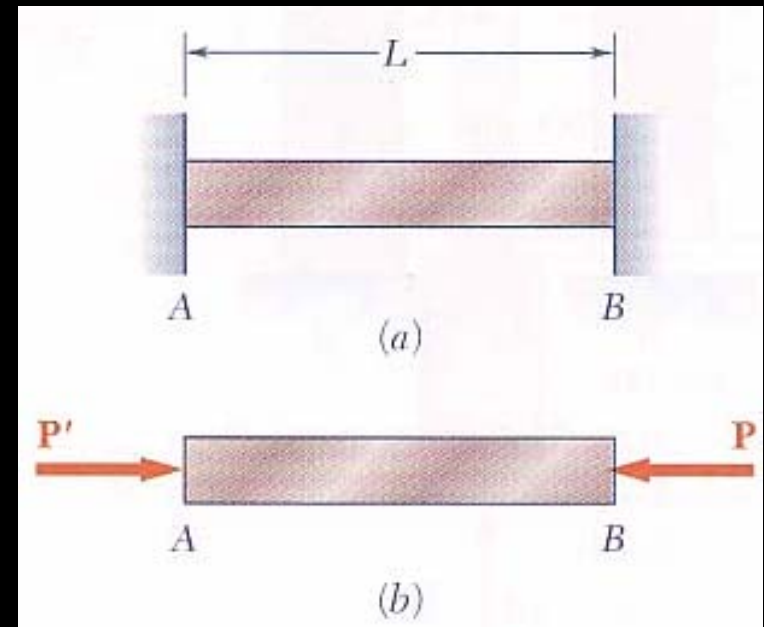
- if thermal movement is restrained stresses are induced

1. bar pushes on supports

2. support pushes back

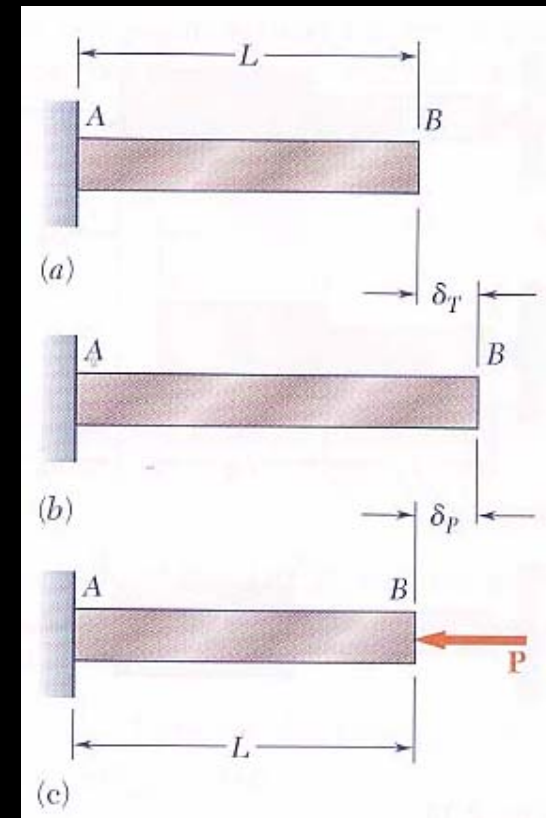
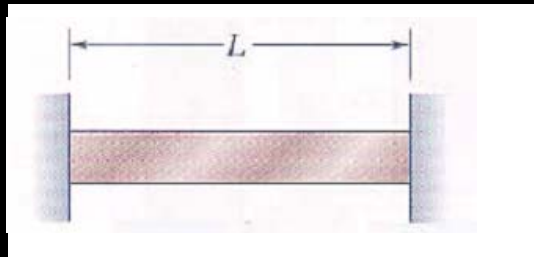
3. reaction causes internal stress

$$f = \frac{P}{A} = \frac{\delta}{L} E$$



# Superposition Method

- can remove a support to make it look determinate
- replace the support with a reaction
- enforce the geometry constraint



# Superposition Method

- total length change restrained to zero

$$\text{constraint: } \delta_P + \delta_T = 0$$

$$\delta_P = -\frac{PL}{AE} \quad \delta_T = \alpha(\Delta T)L$$

$$\text{sub: } -\frac{PL}{AE} + \alpha(\Delta T)L = 0$$

$$f = -\frac{P}{A} = -\alpha(\Delta T)E$$

