Architectural Structures: Form, Behavior, and Design arch 331 Dr. Anne Nichols

SUMMER 2014





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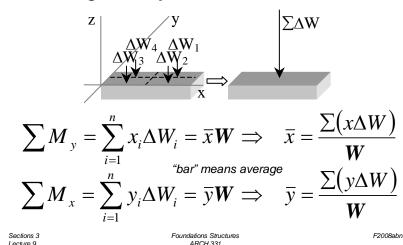
beam sections geometric properties

Sections 1
Lecture 7

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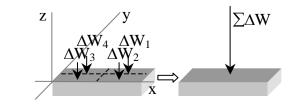
Center of Gravity

• "average" x & y from moment



Center of Gravity

- · location of equivalent weight
- determined with calculus



• sum element weights $W = \int dW$

Sections 2	
Lecture 9	

Sections 4

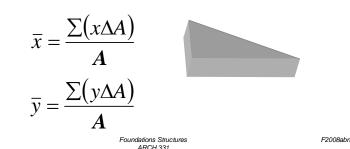
Lecture 9

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Centroid

- "average" x & y of an area
- for a volume of constant thickness

 $-\Delta W = \gamma t \Delta A$ where γ is weight/volume - center of gravity = centroid of area

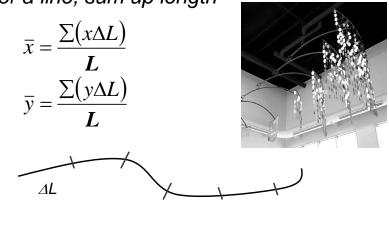


Centroid

Sections 5

Lecture 9

• for a line, sum up length

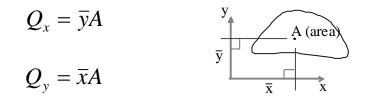


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1st Moment Area

- math concept
- the moment of an area about an axis



Sections 6 Lecture 9

Sections 8

Lecture 9

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Symmetric Areas

- symmetric about an axis
- symmetric about a center point
- mirrored symmetry

Sections 7 Lecture 9

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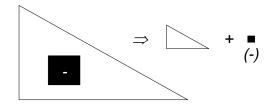
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Composite Areas

- made up of basic shapes
- areas can be negative
- (centroids can be negative for any area)

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Basic Procedure

- 1. Draw reference origin (if not given)
- Divide into basic shapes (+/-) 2.
- Label shapes 3.
- \overline{x} Component | Area $\overline{x}A$ Draw table 4
- Fill in table 5.
 - Σ Sum necessary columns
- 7. Calculate \hat{x} and \hat{y}

Sections 9 Lecture 9

6.

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Moments of Inertia

- 2nd moment area
 - math concept
 - area x (distance)²
- need for behavior of
 - beams
 - columns

FITICAL DISPLACEMENT; Bending BUCKLING L/d = LAPSE stresses NUMBER Torsion Deflections (Excessive deformations) Transverse Loadings

Area Centroids

• Table 7.1 – pg. 242

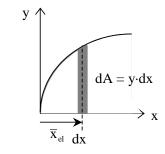


Shape		x	y
Triangular area		b 3 right triangle only	$\frac{h}{3}$
Quarter-circular area	c, C C	$\frac{4r}{3\pi}$	$\frac{4r}{3\pi}$
Semicircular area		0	$\frac{4r}{3\pi}$
Semiparabolic area		$\frac{3a}{8}$	$\frac{3h}{5}$
Parabolic area	$ \begin{array}{c} & & & \\ & & & \\ & & & \\ \hline \\ & & & \\ \hline & & \\ \hline & & & \\ \hline \\ \hline$	0	$\frac{3h}{5}$
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Moment of Inertia

- about any reference <u>axis</u>
- can be negative

$$I_{y} = \int x^{2} dA$$
$$I_{x} = \int y^{2} dA$$



resistance to bending and buckling

Sections 11 Lecture 9

 \overline{y}

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 $\overline{v}A$

Sections 12

Lecture 9

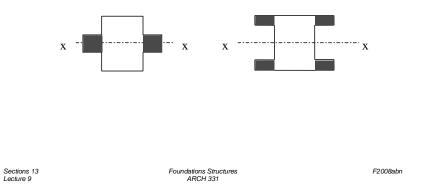
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Moment of Inertia

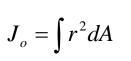
same area moved away a distance

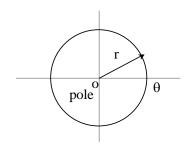
– larger I



Polar Moment of Inertia

- for roundish shapes
- uses polar coordinates (r and θ)
- resistance to twisting



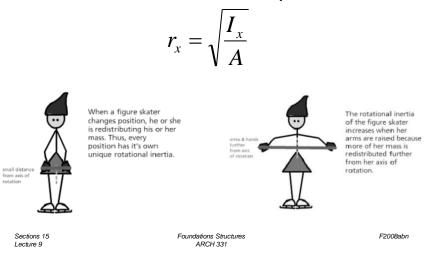




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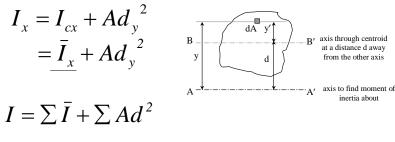
Radius of Gyration

• measure of inertia with respect to area



Parallel Axis Theorem

• can find composite I once composite centroid is known (basic shapes)



$$\bar{I} = I - Ad^2$$

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Basic Procedure

- 1. Draw reference origin (if not given)
- 2. Divide into basic shapes (+/-)
- 3. Label shapes
- 4. Draw table with A, \overline{x} , $\overline{x}A$, \overline{y} , $\overline{y}A$, \overline{I} 's, d's, and Ad²'s
- 5. Fill in table and get \hat{x} and \hat{y} for composite
- 6. Sum necessary columns
- 7. Sum \bar{I} 's and Ad²'s

$$\begin{pmatrix} d_x = \hat{x} - \overline{x} \\ d_y = \hat{y} - \overline{y} \end{pmatrix}$$



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Area Moments of Inertia

• Table 7.2 – pg. 252: (bars refer to centroid)

