#### ARCHITECTURAL STRUCTURES:

FORM, BEHAVIOR, AND DESIGN

ARCH 331

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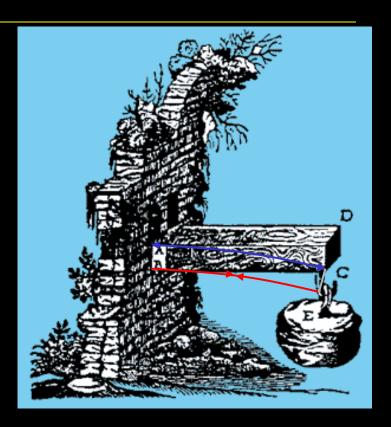
SUMMER 2013

eignt

beams: bending and shear stress

## Beam Bending

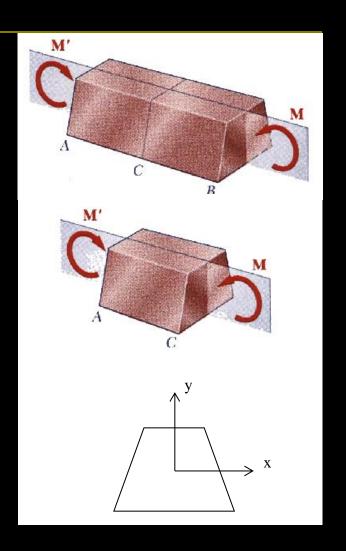
- Galileo
  - relationship between
     stress and depth<sup>2</sup>
- can see
  - top squishing
  - bottom stretching



what are the stress across the section?

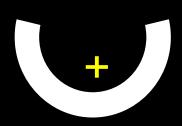
## Pure Bending

- bending only
- no shear
- axial normal stresses from bending can be found in
  - homogeneous materials
  - plane of symmetry
  - follow Hooke's law

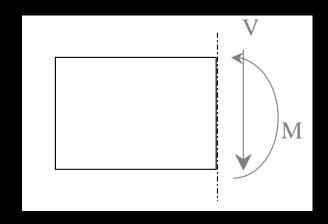


## Bending Moments

sign convention:







• size of maximum internal moment will govern our design of the section

#### Normal Stresses

- geometric fit
  - plane sections remain plane
  - stress varies linearly

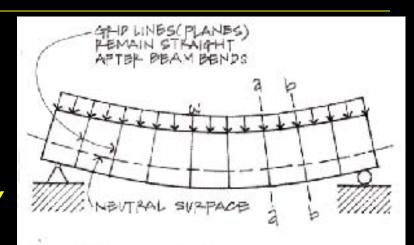


Figure 8.5(b) Beam bending under load.

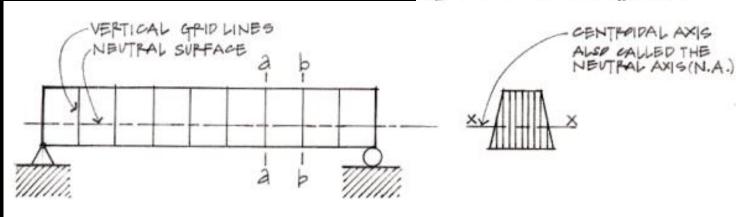


Figure 8.5(a) Beam elevation before loading.

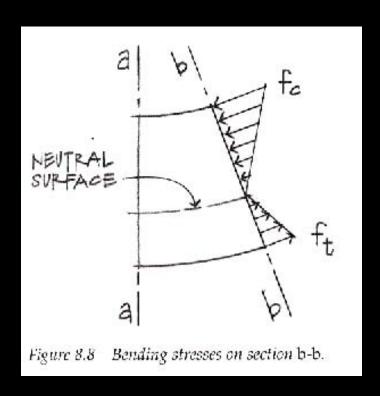
Beam cross section.

#### Neutral Axis

stresses vary linearly

 zero stress occurs at the centroid

 neutral axis is line of centroids (n.a.)

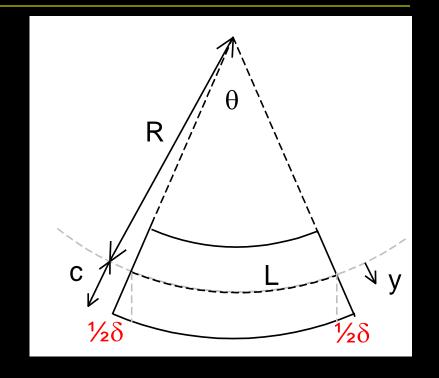


### Derivation of Stress from Strain

pure bending = arc shape

$$L = R\theta$$

$$L_{outside} = (R + y)\theta$$



$$\varepsilon = \frac{\delta}{L} = \frac{L_{outside} - L}{L} = \frac{(R + y)\theta - R\theta}{R\theta} = \frac{y}{R}$$

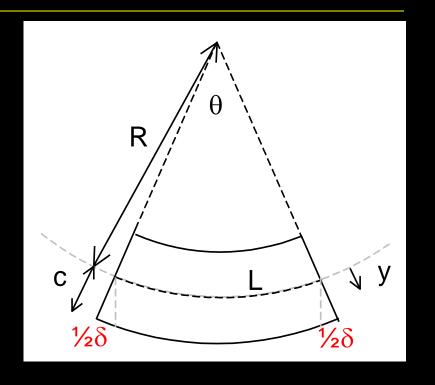
### Derivation of Stress

• zero stress at n.a.

$$f = E\varepsilon = \frac{Ey}{R}$$

$$f_{\text{max}} = \frac{Ec}{R}$$

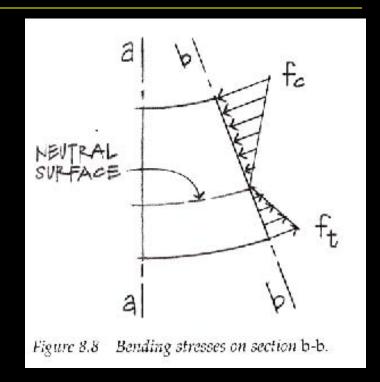
$$f = \frac{y}{c} f_{\text{max}}$$



# Bending Moment

resultant moment from stresses = bending moment!

$$M = \Sigma f y \Delta A$$



$$= \sum \frac{yf_{max}}{c} y \Delta A = \frac{f_{max}}{c} \sum y^2 \Delta A = \frac{f_{max}}{c} I = f_{max} S$$

# Bending Stress Relations

$$\frac{1}{R} = \frac{M}{EI}$$

$$f_b = \frac{My}{I}$$

$$S = \frac{I}{c}$$

curvature

general bending stress

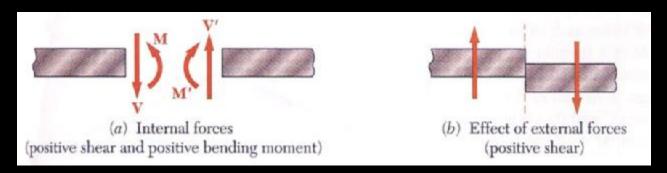
section modulus

$$f_b = \frac{M}{S}$$

$$S_{required} \ge \frac{M}{F_b}$$

required section modulus for design

# Transverse Loading and Shear



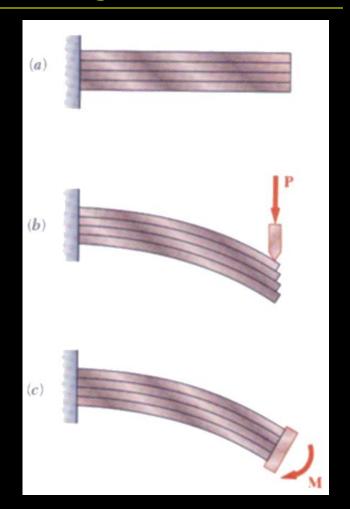
- perpendicular loading
- internal shear
- along with bending moment

# Bending vs. Shear in Design

 bending stresses dominate

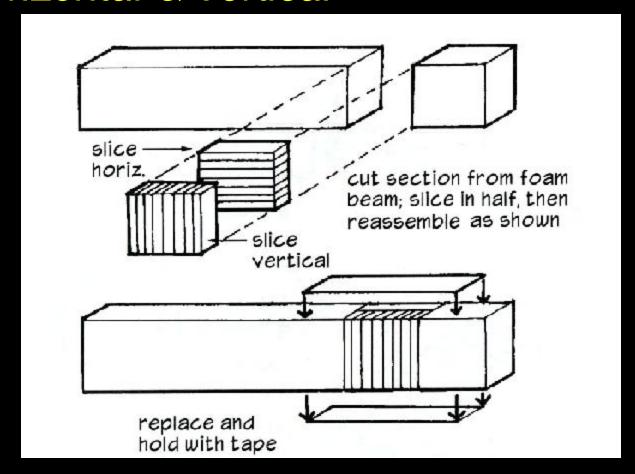
 shear stresses exist horizontally with shear

 no shear stresses with pure bending



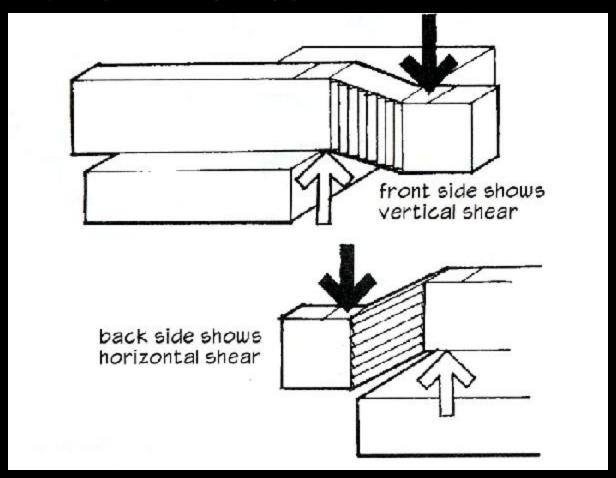
### Shear Stresses

#### horizontal & vertical



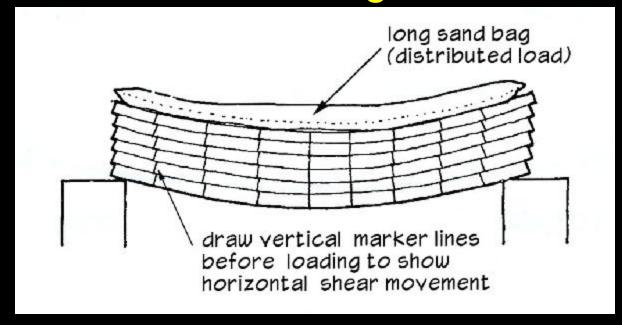
### Shear Stresses

#### horizontal & vertical



#### Beam Stresses

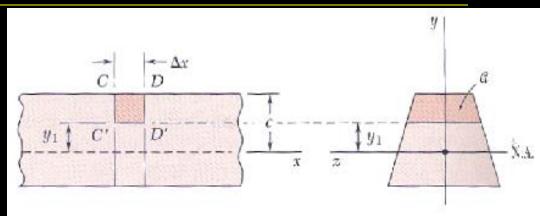
horizontal with bending

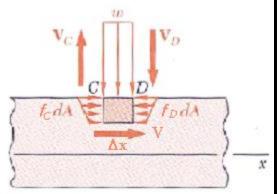


# **Equilibrium**

horizontal force V needed

$$V_{longitudinal} = rac{V_T Q}{I} \Delta x$$



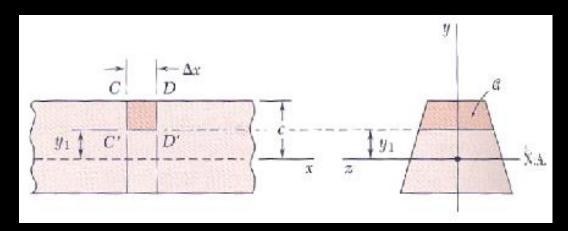


Q is a moment area

### Moment of Area

• Q is a moment area with respect to the n.a. of area <u>above or below</u> the horizontal

 Q<sub>max</sub> at y=0 (neutral axis)



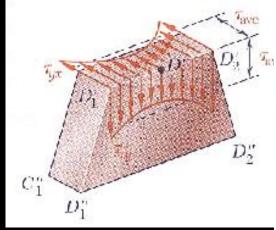
q is shear flow:

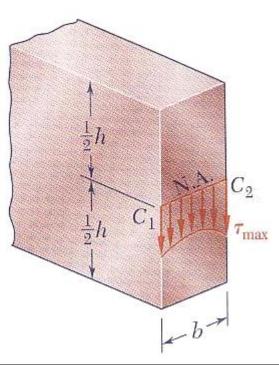
$$q = rac{V_{longitudinal}}{\Delta x} = rac{V_{T}Q}{I}$$

# Shearing Stresses

$$f_{v} = \frac{V}{\Delta A} = \frac{V}{b \cdot \Delta x}$$

$$f_{v-ave} = \frac{VQ}{Ib}$$





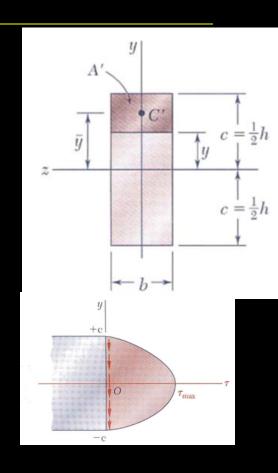
- $f_{v-ave} = 0$  on the top/bottom
- b min may not be with Q max
- with  $h/4 \ge b$ ,  $f_{v-max} \le 1.008 f_{v-ave}$

# Rectangular Sections

$$I = \frac{bh^3}{12} \qquad Q = A\bar{y} = \frac{bh^2}{8}$$

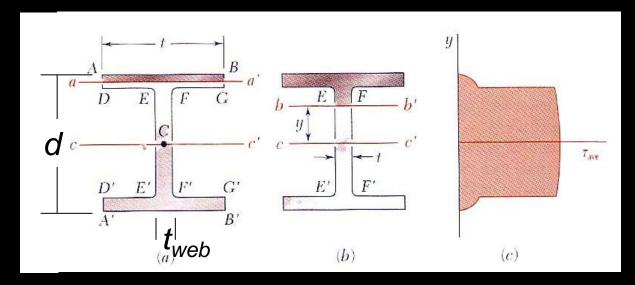
$$f_v = \frac{VQ}{Ib} = \frac{3V}{2A}$$

•  $f_{v-max}$  occurs at <u>n.a.</u>



#### Steel Beam Webs

- W and S sections
  - b varies

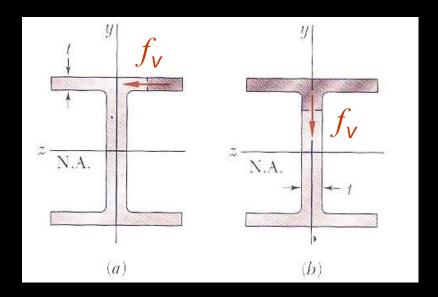


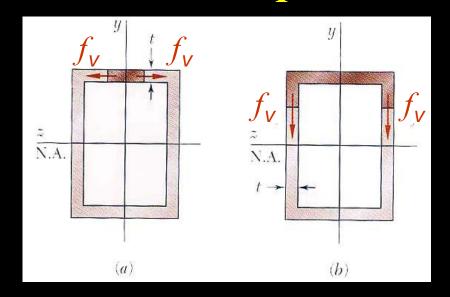
- stress in flange negligible
- presume constant stress in web

$$f_{v-\text{max}} = \frac{3V}{2A} \approx \frac{V}{A_{web}}$$

### Shear Flow

- loads applied in plane of symmetry
- cut made perpendicular  $q = \frac{VQ}{q}$

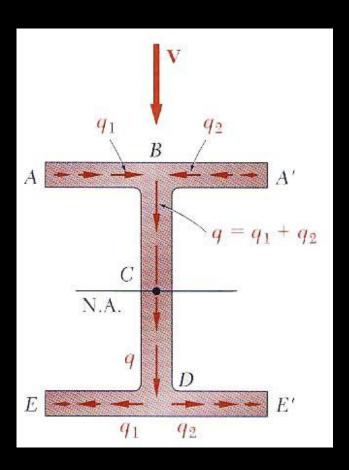




# Shear Flow Quantity

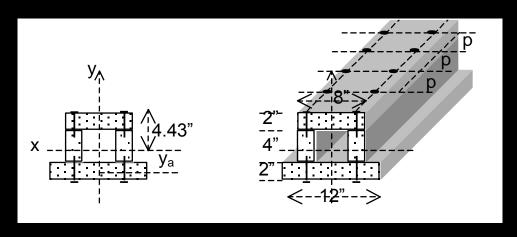
sketch from Q

$$q = \frac{VQ}{I}$$



# Connectors Resisting Shear

- plates with
  - nails
  - rivets
  - bolts
- splices



$$rac{V_{longitudinal}}{p} = rac{VQ}{I}$$
 $nF_{connector} \geq rac{VQ_{connected\ area}}{I} \cdot p$ 

### Vertical Connectors

isolate an area with vertical interfaces

$$nF_{connector} \ge \frac{VQ_{connected\ area}}{I} \cdot p$$

# Unsymmetrical Shear or Section

- member can bend and twist
  - not symmetric
  - shear not in that plane
- shear center
  - moments balance

